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MULTIVARIABLE CONTROL OF A BOILER - AN APPLICATION OF LINEAR  
QUADRATIC CONTROL THEORY <sup>†</sup>

K. Eklund

ABSTRACT

An application of linear quadratic control theory to a multi-variable system is presented. The process is a boiler and the object of the control is to keep the drum pressure and the drum level constant when the load changes. The load disturbances were modelled from measurements as a stationary stochastic process with rational spectral density function. The crucial difficulty when using optimal theory for design is to find the parameters of the loss functional. A method for choosing these parameters is outlined. A method to eliminate steady state errors is also presented. A Kalman filter for the estimation of the state vector as well as the load disturbance was included. The control situation was simulated on a hybrid computer. The results of these simulations as well as core memory requirements and execution time for the control algorithm are given.

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## 1. INTRODUCTION

This report presents an application of linear quadratic control theory to a multivariable system. The process used is a simplified boiler which can be described with a linear constant coefficient dynamical system of the 5:th order. The process has three inputs and two outputs. There are considerable interactions between the inputs and outputs of the process. For such processes conventional synthesis methods are not very attractive. However, using linear quadratic control theory we can synthesize multivariate control laws in a systematic manner.

The boiler control problem is to keep the process output variables constant when the process is disturbed. This type of control problem is very common in process industries. Particular attention must be paid to formulate the control problem as an optimization problem.

A hybrid computer was used to simulate the control situation. This is as far as the control law implementation is concerned quite realistic. The process, however, will be the model equations simulated on the analog computer.

In section 2 we give a résumé of the linear quadratic control theory. The theory requires that we have models of the process and the disturbances available. In section 3 a short presentation of the model of the boiler is given and the models of the disturbances are derived and discussed in section 4. A method to eliminate steady state errors using the technique of feedforward is presented in section 5. It is also shown that the combination of a feedforward and a Kalman filter is equivalent to the introduction of an integrator. The crucial difficulty when using optimal control theory for design is to find the parameters of the loss functional. In section 6 we discuss this problem and outline a method for choosing these parameters. The sampling interval affects the quality of control which decrease with increasing length of the sampling interval. The choice of the sampling interval is discussed in section 7. The complete control law is given in section 8. In this section we also discuss the sensitivity of the Kalman filter to changes of the process parameters. In section 9

we give the core memory requirements and execution time for the control algorithm. The scaling problems which arise when we use fix point arithmetic are discussed. The results of analog and hybrid simulations are given in section 10.

## 2. RÉSUMÉ OF LINEAR QUADRATIC CONTROL THEORY

The theory can be developed both in the continuous and the discrete case. The résumé given here is restricted to the continuous case. In the discrete case the differential equations are replaced by difference equations but the structure of the solution is identical.

Consider the linear system

$$\begin{aligned} \frac{dx(t)}{dt} &= A(t) x(t) + B(t) u(t) + w_1(t) \\ y(t) &= C(t) x(t) + w_2(t) \end{aligned} \quad (2.1)$$

for  $t_0 \leq t < \infty$ .  $x(t)$  is the state  $n$ -vector,  $u(t)$  is the control  $m$ -vector and  $y(t)$  is the output  $k$ -vector.

The formal expression (2.1) can be interpreted as a stochastic differential equation in the usual manner. Since we will not use (2.1) for any analysis we use this formal expression instead of the mathematically rigorous but more elaborate notions of stochastic differential equations.

The elements of the matrices  $A(t)$ ,  $B(t)$  and  $C(t)$  are continuous and bounded functions of  $t$ . The variables  $w_1(t)$  and  $w_2(t)$  are white noise with zero mean and the covariance functions

$$\begin{aligned} E w_1(t) w_1^T(t+\tau) &= R_1(t) \delta(\tau) \\ E w_2(t) w_2^T(t+\tau) &= R_2(t) \delta(\tau) \end{aligned} \quad (2.2)$$

where  $\delta(\tau)$  is the Dirac measure.  $R_1(t)$  is a symmetric nonnegative definite matrix and  $R_2(t)$  is a symmetric positive definite matrix. The elements of  $R_1(t)$  and  $R_2(t)$  are continuous and bounded functions of  $t$ . The initial state is a random variable with

$$\begin{aligned} E x(t_0) &= m \\ \text{cov } x(t_0) x^T(t_0) &= R_0 \end{aligned} \quad (2.3)$$

The object of the control is to minimize the loss functional

$$V(x_0, t_0, t_1, u) = E \left\{ x^T(t_1) Q_0 x(t_1) + \int_{t_0}^{t_1} [x^T(s) Q_1(s) x(s) + u^T(s) Q_2(s) u(s)] ds \right\} \quad (2.4)$$

where  $Q_0$  and  $Q_1(t)$  are symmetric nonnegative definite matrices and  $Q_2(t)$  is a symmetric positive definite matrix. The parameter  $t_1$  may be infinite. The elements of  $Q_1(t)$  and  $Q_2(t)$  are continuous and bounded functions of  $t$ .

The solution of this problem can be separated into two independent problems: 1) a deterministic control problem and 2) an estimation problem.

The solution of the deterministic control problem is given by

$$u(t) = -L(t) \hat{x}(t) \quad (2.5)$$

where  $\hat{x}(t)$  is the estimated state vector and

$$L(t) = Q_2^{-1}(t) B^T(t) S(t; t_1) \quad (2.6)$$

$S(t; t_1)$  is the solution of a Riccati equation. This equation depends on  $A(t)$ ,  $B(t)$ ,  $Q_0$ ,  $Q_1(t)$ ,  $Q_2(t)$  but does not depend on  $C(t)$ ,  $R_0$ ,  $R_1(t)$ ,  $R_2(t)$ .

The minimum mean square estimate is given by

$$\frac{d\hat{x}(t)}{dt} = A(t)\hat{x}(t) + B(t)u(t) + K(t)[y(t) - C(t)\hat{x}(t)] \quad (2.7)$$

where

$$K(t) = P(t; t_0) C^T(t) R_2^{-1}(t) \quad (2.8)$$

The matrix  $P(t; t_0)$  is the solution of a Riccati equation which depends on  $A(t)$ ,  $C(t)$ ,  $R_0$ ,  $R_1(t)$  and  $R_2(t)$ .

The deterministic control problem and the estimation problem are dual and the feedback matrix  $L(t)$  and the filter gain matrix  $K(t)$  can be computed using the same algorithm. If the time point  $t_1$  is set equal to infinity we will obtain the stationary values of  $L(t)$  and  $K(t)$ . There are no constraints on the state vector  $x(t)$  and the control vector  $u(t)$ . In the time invariant case the closed system will be stable if (2.1) is controllable and observable and if the pair of matrices  $(Q_1, A)$  and  $(R_1, A^T)$  are observable. A detailed presentation of the theory is found in {1}, {5}.

### 3. BOILER MODEL

We consider a drum boiler with natural circulation. The configuration is given in Fig. 1.

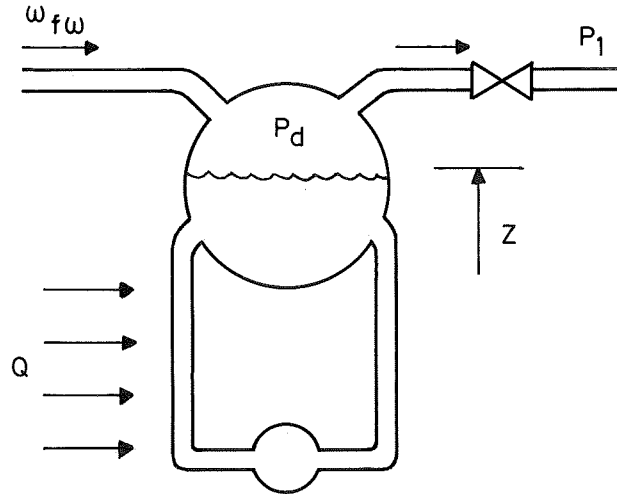


Fig. 1 - Simplified boiler configuration

We use a detailed model only for the drum-downcomer-riser loop of the boiler. The superheaters are simulated with a restriction only.

The linearized model on standard form is

$$\begin{aligned} \frac{dx(t)}{dt} &= Ax(t) + Bu(t) + Fv_1(t) \\ y(t) &= Cx(t) + v_2(t) \end{aligned} \quad (3.1)$$

where A,B,F and C are constant matrices. It is a fifth order model and the state variables are

- $x_1(t)$  drum pressure  $p_d$
- $x_2(t)$  drum liquid level  $z$
- $x_3(t)$  drum liquid temperature
- $x_4(t)$  riser wall temperature
- $x_5(t)$  steam quality

The control variables are

- $u_1(t)$  heat flow to the risers  $Q$
- $u_2(t)$  feedwater flow  $w_{fw}$



and the output variables are

$y_1(t)$       the measured drum pressure  
 $y_2(t)$       the measured drum level.

The disturbances are

$v_1(t)$       load changes  $p_1$   
 $v_2(t)$       measurement noise

The heat input variable is the heat flow to the risers and not the fuel flow. The feedwater enthalpy is taken as a constant and not as an input variable. In a power station boiler the pressure  $p_1$  is the pressure before the throttle valve of the turbine. Changes in the demand for steam will instantaneously cause changes in this pressure. Thus we can use the pressure  $p_1$  as a direct measure of the load changes. The controlled variables are the output variables and the object of control is to keep these variables constant when the load changes. A detailed discussion of the model is found in {2}.

Numerical values of the matrices A, B, C and F used in this report are found in Appendix A. The values apply to a power station boiler with a maximum steam flow of about 350 t/h. The drum pressure is 140 bar. The operating point is 90% of full load. The eigenvalues of the matrix A are

$-5.99 \cdot 10^{-2} \pm 1.72 \cdot 10^{-2} i$   
 $-1.81 \cdot 10^{-1}$   
 $-8.59 \cdot 10^{-2}$   
0.00

It is not easy to give a simple physical interpretation of the eigenvalues because of the interaction in the system. Notice that the second column of A equals zero which gives a zero eigenvalue. This also means that the second state variable is not coupled to the other state variables.

The simulated responses of the state variables to step changes in the three input variables are given in Fig. 2. Notice the non-minimum phase characteristics of the drum level and steam quality responses. These two state variables are closely related. The step responses also show that we have a considerable interaction in the process.

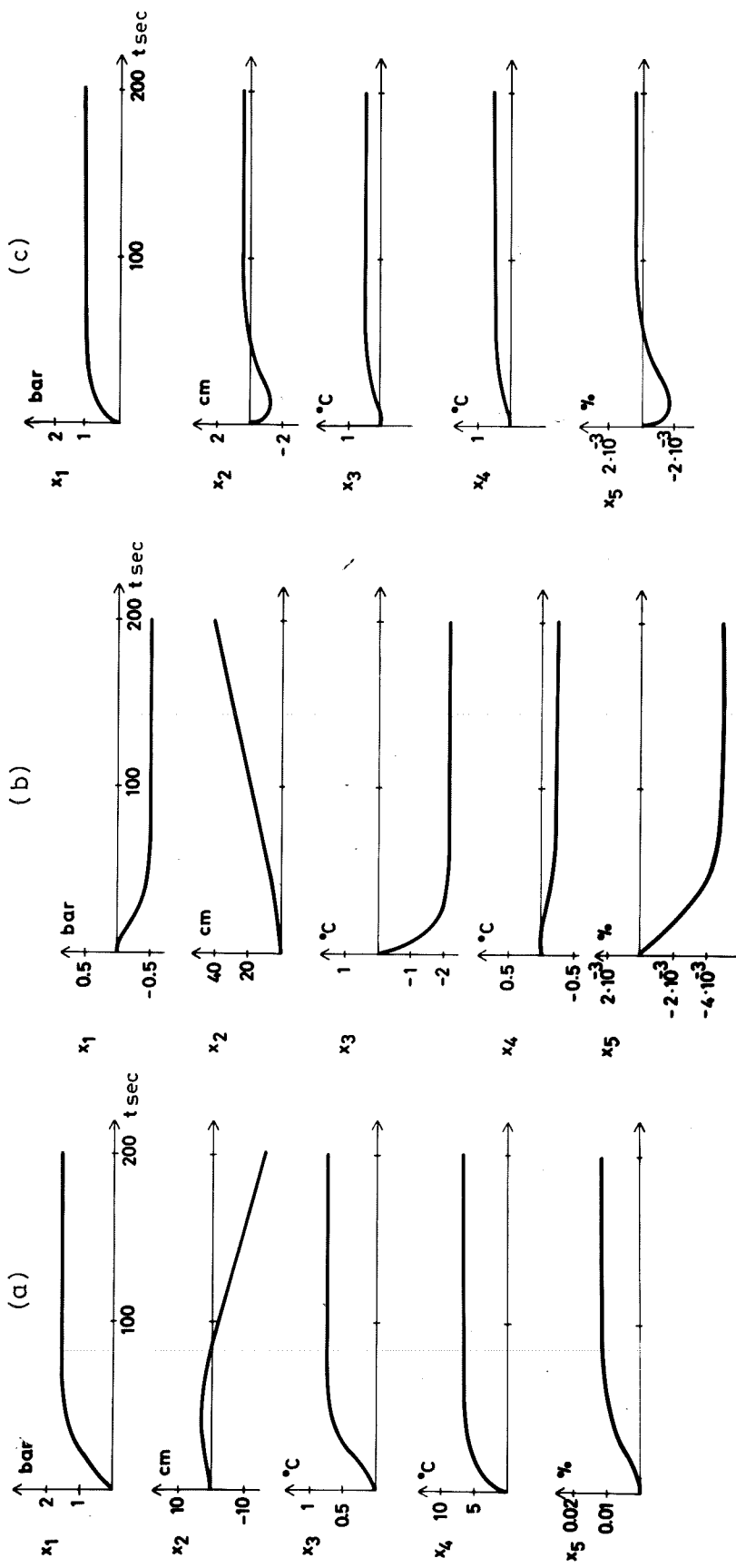


Fig. 2 - Responses of state variables to a step change in (a) heat flow to the risers (b) feedwater flow and (c) pressure  $P_1$

#### 4. CHARACTERISTICS OF DISTURBANCES

In section 2 it was stated that the solution of the formulated problem requires that we know the characteristics of the random processes involved. In the boiler application the input noise is the load disturbance  $v_1(t)$  and the measurement noise is  $v_2(t)$ . (See equation (3.1)). In this section we will give the characteristics of the random processes which are used to describe these noises.

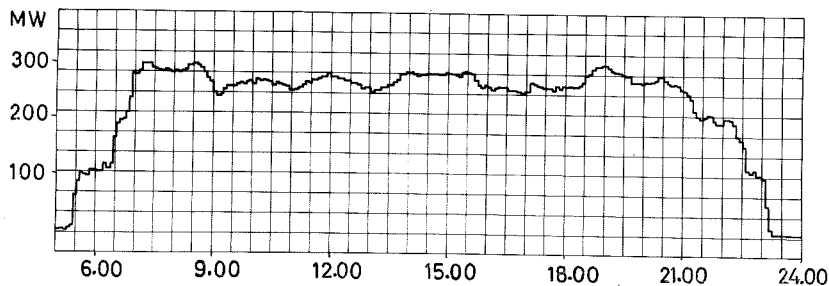


Fig. 3 - The power generated by four hydroelectric power stations in the time interval 5<sup>00</sup>-24<sup>00</sup> a weekday

Fig. 3 shows the power generated by four hydroelectric power stations in the north of Sweden a weekday. The sampling interval of the measurement is 300 sec. The load changes during 5<sup>00</sup>-7<sup>00</sup> and 21<sup>00</sup>-24<sup>00</sup> a'clock are for the main part ordered load changes. During the time interval 7<sup>00</sup>-21<sup>00</sup> the variations are mostly due to the control of the mains frequency. Since the dynamics of a hydroelectric power station are fast compared to the dynamics of a thermal power station we will use the recording in the interval 7<sup>00</sup>-21<sup>00</sup> as a measurement of the demand for power.

A set of measurements for various weekdays have been used to determine the parameters of a model of the load disturbance. We assume that the disturbance is a stationary process with rational spectrum. Such a process can always be represented with a linear model

$$A(z^{-1})y(t) = \lambda C(z^{-1})e(t) \quad (4.1)$$

where  $e(t)$  is a sequence of independent normal (0,1) random variables.  $z$  is the shift operator

$$z y(t) = y(t + T)$$

and  $T$  is the sampling interval.  $A(z^{-1})$  and  $C(z^{-1})$  are polynomials in the inverse shiftoperator  $z^{-1}$ . The identification method used is the maximum likelihood method. A presentation of the used method is found in [4]. The identification gives a first order system

$$y(t) = \lambda \frac{1 + c_1 z^{-1}}{1 + a_1 z^{-1}} e(t) \quad (4.2)$$

where the average values of the coefficients and standard deviations were

$$\begin{aligned} a_1 &= -0.92 \pm 0.036 \\ c_1 &= 0.10 \pm 0.071 \\ \lambda &= 5.6 \end{aligned}$$

The coefficient  $c_1$  is quite small and roughly zero within one standard deviation. We will therefore assume that  $c_1$  equals zero. This is not a severe assumption and will simplify the computations. The variable  $y(t)$  has the dimension MW and gives the deviation from the mean value of the generated power. The mean value is about 275 MW. To fit the boiler model we must find the equivalence between  $y(t)$  and the pressure  $v_1(t)$ . If we also consider that the maximum power generated by the studied boiler is about 125 MW we get

$$v_1(t) = \lambda \cdot \frac{1}{1 + a z^{-1}} e(t) \quad (4.3)$$

where

$$\begin{aligned} a &= -0.92 \\ \lambda &= 0.225 \end{aligned}$$

For the analog and hybrid simulations it is convenient to have a continuous approximation of the load disturbance model. Assume a first order continuous system

$$\begin{aligned} \frac{dx(t)}{dt} &= \alpha x(t) + \mu \omega(t) \\ v_1(t) &= x(t) \end{aligned} \quad (4.4)$$

where  $\omega(t)$  is white noise with zero mean and the covariance function

$$\text{cov } \omega(t) \omega(t + \tau) = \delta(\tau)$$

The covariance functions for the discrete (4.3) and the continuous (4.4) representation of  $v_1(t)$  are

$$r(n) = a^n \cdot \frac{\lambda^2}{1 - a^2} \quad (4.5)$$

$$r(\tau) = \frac{\mu^2}{2|\alpha|} \cdot e^{-|\alpha||\tau|} \quad (4.6)$$

Equating the covariance functions (4.5) and (4.6) for  $n = 0$ ,  $\tau = 0$  and  $n = 1$ ,  $\tau = 300$  sec we get

$$\alpha = -2.78 \cdot 10^{-4}$$

$$\mu = 0.0130$$

The continuous system (4.4) has a time constant  $T$  of 3600 sek. For our purpose we will use an observation time less than 2000 sec. During this time the system (4.4) will practically act as if  $\alpha$  was equal to zero. With this approximation we get

$$\frac{dx(t)}{dt} = \mu \omega(t)$$

$$v_1(t) = x(t) \quad (4.7)$$

Spectral density functions for the load disturbances generated by equations (4.4) and (4.7) are given in Fig. 4. The spectral density function of  $\omega(t)$  equals a constant  $A$ .

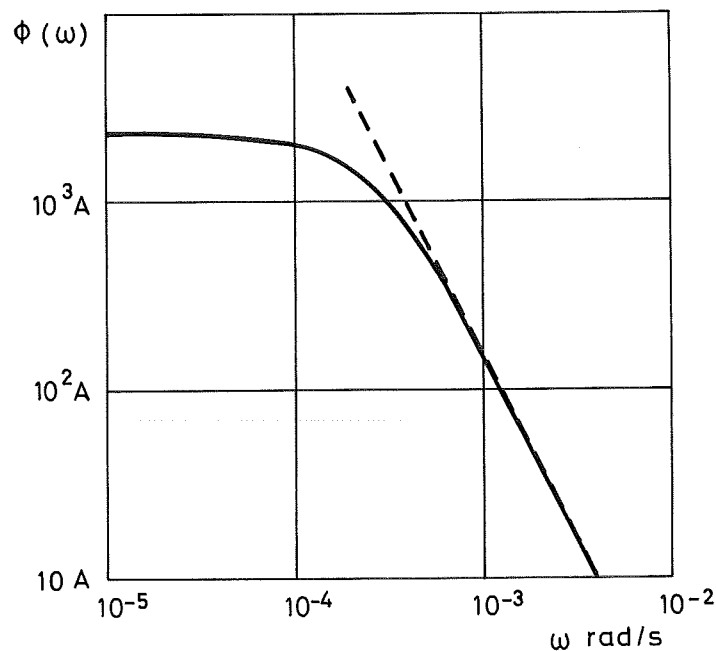


Fig. 4 - Spectral densities for the load disturbance generated by equation (4.4) (—) and equation (4.7) (----)

The frequency content above  $\omega = 10^{-3}$  rad/s is very small. This is expected since the sampling interval in the original measurement was 300 sec. However, this is a reasonable noise considering the dynamics of the boiler.

No recordings of the measurement noise of the drum pressure and drum level signals were available. We will therefore assume that the measurement noises are pure random processes and that the amplitude distributions are normal with zero mean. The choice of the standard deviations are discussed in section 8.

### 5. ELIMINATION OF STEADY STATE ERRORS

When the control law (2.5) given in section 2 is used, there will be no steady state errors if the disturbance is an initial error in any state variable. But we also require that the steady state errors of the controlled variables are zero after e.g. a step change of the disturbance  $v(t)$ , see equation (5.1). To achieve this, we will use the technique of feedforward.

We will first consider the system

$$\frac{dx(t)}{dt} = Ax(t) + Bu(t) + Fv(t) \quad (5.1)$$

where  $v(t)$  is a  $s$ -vector. We assume that the state vector  $x(t)$  and the disturbance vector  $v(t)$  can be measured directly. Using index  $o$  to indicate steady state values equation (5.1) gives

$$Ax_o + Bu_o + Fv_o = 0 \quad (5.2)$$

We thus find that with a control law  $u_o = -Lx_o$  there will in general be a steady state error. To eliminate this we add a feedforward term from the disturbance  $v(t)$  to the stationary control law. Hence

$$u_o = -Lx_o - Rv_o \quad (5.3)$$

where  $R$  is a constant matrix which will be chosen in such a way that the steady state values of  $i$  components of the state vector are zero. We assume that equation (5.1) is arranged so that these components are the first  $i$  components. Introduce the notations

$$\bar{A} [n \times (n-i)] = [a_{i+1} \dots a_n]$$

$$\bar{L} [m \times (n-i)] = [\ell_{i+1} \dots \ell_n]$$

$$\bar{x} = [x_{i+1} \dots x_n]^T$$

where  $a_k$  and  $\ell_k$  stand for the  $k$ :th column of  $A$  and  $L$  respectively. Introducing the zero error requirement in equation (5.2) and (5.3) we get

$$\bar{A}x_o + Bu_o + Fv_o = 0 \quad (5.4)$$

$$u_o = -\bar{L}x_o - Rv_o \quad (5.5)$$

or

$$[\bar{A}|B] \begin{bmatrix} - \\ \underline{x}_0 \\ - \\ \underline{v}_0 \end{bmatrix} = - F \underline{v}_0 \quad (5.6)$$

$$R \underline{v}_0 = - [\bar{L}|I] \begin{bmatrix} - \\ \underline{x}_0 \\ - \\ \underline{u}_0 \end{bmatrix} \quad (5.7)$$

The existence of a solution to equation (5.6) determines possible numbers  $i$ . For example if  $i$  equals the number of control variables a unique solution to equation (5.6) exists for all matrices  $F$ , if the inverse of  $[\bar{A}|B]$  exists. If the inverse does not exist, we must require that the columns of  $F$  lie in the column space of  $[\bar{A}|B]$ . In this case the solution is obtained using the pseudo-inverse of  $[\bar{A}|B]$ . If equation (5.6) has a solution the feedforward matrix  $R$  is computed using equation (5.7).

In many physical systems the number  $i$  will equal the number of control variables. For this case it has been shown numerically for several specific problems that the feedforward matrix  $R$  obtained when using the technique described above can be obtained directly when the control law is computed. The details of this is given in Appendix B.

We will now consider the case when only the output vector  $y(t)$  can be measured. The system then is

$$\frac{dx(t)}{dt} = Ax(t) + Bu(t) + Fv(t)$$

$$y(t) = Cx(t) + \omega_2(t) \quad (5.8)$$

We assume that the disturbance  $v(t)$  is a Wiener process. Hence

$$\frac{dv(t)}{dt} = \omega_1(t) \quad (5.9)$$

$\omega_1(t)$  and  $\omega_2(t)$  are white noise with zero mean and covariance functions given by equation (2.2). Especially the equations (5.8) and (5.9) hold for the boiler application. Combining equations (5.8) and (5.9) the system equations get the form of equation (2.1)

$$\frac{d}{dt} \begin{bmatrix} \underline{x}(t) \\ \underline{v}(t) \end{bmatrix} = \begin{bmatrix} A & F \\ 0 & 0 \end{bmatrix} \begin{bmatrix} \underline{x}(t) \\ \underline{v}(t) \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} u(t) + \begin{bmatrix} 0 \\ I \end{bmatrix} \omega_1(t)$$

$$y(t) = Cx(t) + \omega_2(t) \quad (5.10)$$



Equation (5.10) is used when the Kalman filter is computed. The filter equation gives the estimates of the state and disturbance vectors. The control law then is

$$u(t) = - L\hat{x}(t) - R\hat{v}(t) \quad (5.11)$$

The combination of a Kalman filter and a feedforward is equivalent to the introduction of an integrator, if the disturbance  $v(t)$  is a Wienerprocess. To illustrate this we will consider an example.

Example

Consider a first order system with one control and one disturbance variable

$$\frac{dx_1(t)}{dt} = u(t) + v(t)$$

$$y(t) = x_1(t) + \omega_2(t) \quad (5.12)$$

The disturbance  $v(t)$  is a Wienerprocess

$$\frac{dx_2(t)}{dt} = \omega_1(t)$$

$$v(t) = x_2(t) \quad (5.13)$$

The control law including the feedforward is

$$u(t) = - \ell\hat{x}_1(t) - \hat{v}(t)$$

Combining equations (5.12) and (5.13) we get

$$\frac{dx(t)}{dt} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x(t) + \begin{bmatrix} 1 \\ 0 \end{bmatrix} u(t) + \begin{bmatrix} 0 \\ 1 \end{bmatrix} \omega_1(t)$$

The filter equation then is

$$\frac{d\hat{x}(t)}{dt} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} \hat{x}(t) + \begin{bmatrix} 1 \\ 0 \end{bmatrix} u(t) + \begin{bmatrix} k_1(t) \\ k_2(t) \end{bmatrix} \{y(t) - \hat{x}_1(t)\}$$

Using the stationary filter gains  $k_1$  and  $k_2$  we can compute the transfer function of the feedback loop. We get

$$U(s) = G(s) Y(s)$$

where

$$G(s) = - \frac{(\ell k_1 + k_2)s + \ell k_2}{s(s + \ell + k_1)}$$

Since  $G(s)$  contains an integrator the steady state error of  $y(t)$  equals zero.

Notice that if any of the conditions, a Wienerprocess disturbance or correct feedforward, are violated there will be a steady state error. It is easy to verify that components of the state vector which have no steady state error are stationary processes. The feedforward does not influence the dynamics of the closed system and thus not the guaranteed stability associated with the optimal feedback. It should also be mentioned that this technique to compute the feedforward matrix  $R$  and the properties discussed above also apply in the discrete case.

## 6. CHOICE OF LOSS FUNCTIONAL

The feedback matrix  $L(t)$  given by equation (2.6) does not depend on the disturbances but only on the loss functional which determines the control law uniquely. Hence we will consider the system equation

$$\frac{dx(t)}{dt} = Ax(t) + Bu(t) \quad (6.1)$$

and the loss functional

$$V = \frac{1}{2} x^T(t_1) Q_0 x(t_1) + \frac{1}{2} \int_{t_0}^{t_1} \{ \alpha x^T(s) Q_1 x(s) + u^T(s) Q_2 u(s) \} ds \quad (6.2)$$

where the scalar  $\alpha$  is used to vary the weight of the state variables in relation to the control variables.

The interpretation of  $Q_0$ ,  $Q_1$ ,  $Q_2$  is apparent.  $Q_0$  represents the weight we put on the difference between the reached and the desired state at the terminal time  $t_1$ .  $Q_1$  and  $Q_2$  represents how we weight deviations from the desired state of the state vector  $x(t)$  and the use of the control vector  $u(t)$  in the control interval.

If we only use the diagonal elements of the loss functional matrices it is easy to qualitatively predict the effect of a parameter change on the closed loop dynamics. For example if we increase the  $ii$ -th element of  $Q_1$  the deviations from the desired state of the state variable  $x_i$  will decrease. Since the relative weight of all other state variables then is decreased, the deviations in these variables will increase. If all the elements of  $Q_1$  are increased the poles of the closed system will move to the left in the complex  $s$ -plane and the system becomes faster. At the same time the magnitude of the control variables will increase.

In the boiler application we will use the stationary value of the feedback matrix  $L(t)$ . This is physically motivated since in the control problem defined for the boiler the terminal time  $t_1$  can be regarded as plus infinity. This also means that  $Q_0$  can be set equal to zero.

In many cases there is no rational a priori choice of the parameters of the loss functional. Especially there is no rational way to match the relative magnitudes of  $Q_1$  and  $Q_2$ . To find the

parameters of the loss functional we will use the iteration procedure shown in Fig. 5.

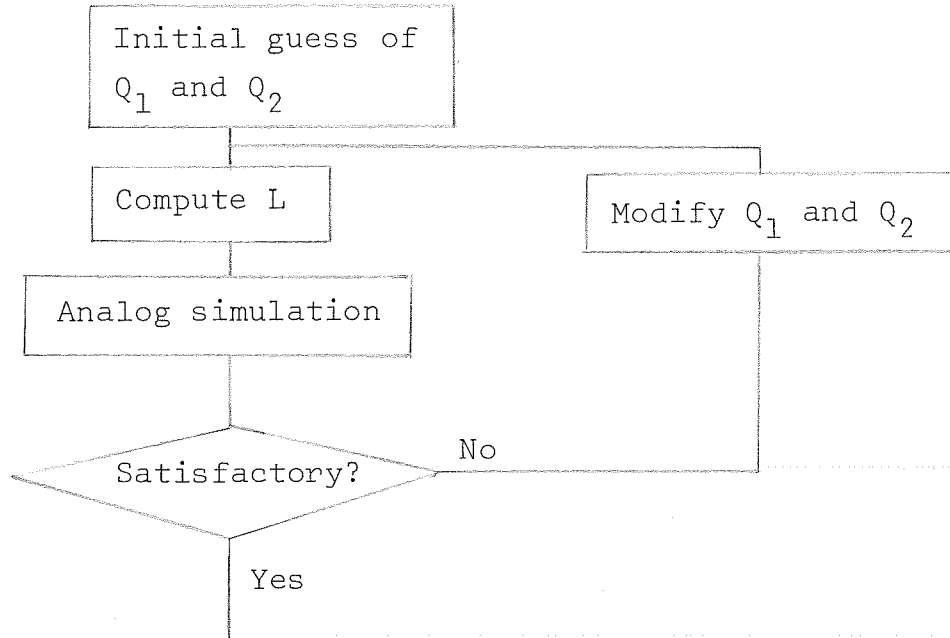


Fig. 5 - The iteration procedure for finding the loss functional matrices.

The idea behind the initial guess of  $Q_1$  and  $Q_2$  is to give all punished variables the same weight in the loss functional. This is achieved by normalizing the variables with an assumed maximum deviation. Notice that we have to punish all control variables but not all state variables since  $Q_2$  must be positive definite and  $Q_1$  only nonnegative definite. The control law is computed and evaluated by simulation. We can not take any constraints on the control vector explicitly into account. We thus have to balance a fast response of the closed system against the magnitude of the control variables for typical disturbances. It is important that the feedforward term is included in the simulations since this term alters the magnitude of the control variables. Notice, however, that we can not change the steady state value of the control vector by a change of a parameter of the loss functional.

The object of the control in the boiler application is to keep the drum pressure  $x_1(t)$  and the drum level  $x_2(t)$  constant with no steady state error. This error can be eliminated since the inverse of the matrix  $[\bar{A}|B]$  exists in this case. The feedforward matrix R is computed using equations (5.6) and (5.7).

The initial guess of  $Q_1$  and  $Q_2$  is

$$Q_1 = \begin{bmatrix} \left(\frac{1}{x_{1\max}}\right)^2 & 0 & 0 & 0 & 0 \\ 0 & \left(\frac{1}{x_{2\max}}\right)^2 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

and

$$Q_2 = \begin{bmatrix} \left(\frac{1}{u_{1\max}}\right)^2 & 0 \\ 0 & \left(\frac{1}{u_{2\max}}\right)^2 \end{bmatrix}$$

where the assumed maximum deviations are

$$\begin{aligned} x_{1\max} &= 10 \quad \text{bar} \\ x_{2\max} &= 0.1 \quad \text{m} \\ u_{1\max} &= 10^4 \quad \text{kJ/s} \\ u_{2\max} &= 10 \quad \text{kg/s} \end{aligned}$$

The disturbances used in the analog simulation are a 10% step change of the load and a disturbance in the initial value of  $x_1(t)$  and  $x_2(t)$ .

Fig. 6 and 7 give the results of the simulation using the initially guessed  $Q_1$  and  $Q_2$  with  $\alpha = 10$ . The corresponding control law will be called control law I. The responses of the controlled variables  $x_1(t)$  and  $x_2(t)$  are not satisfactory. Especially the response of  $x_2(t)$  is quite slow. Fig. 7 shows the responses to a load change with and without the feedforward matrix included in the control law.

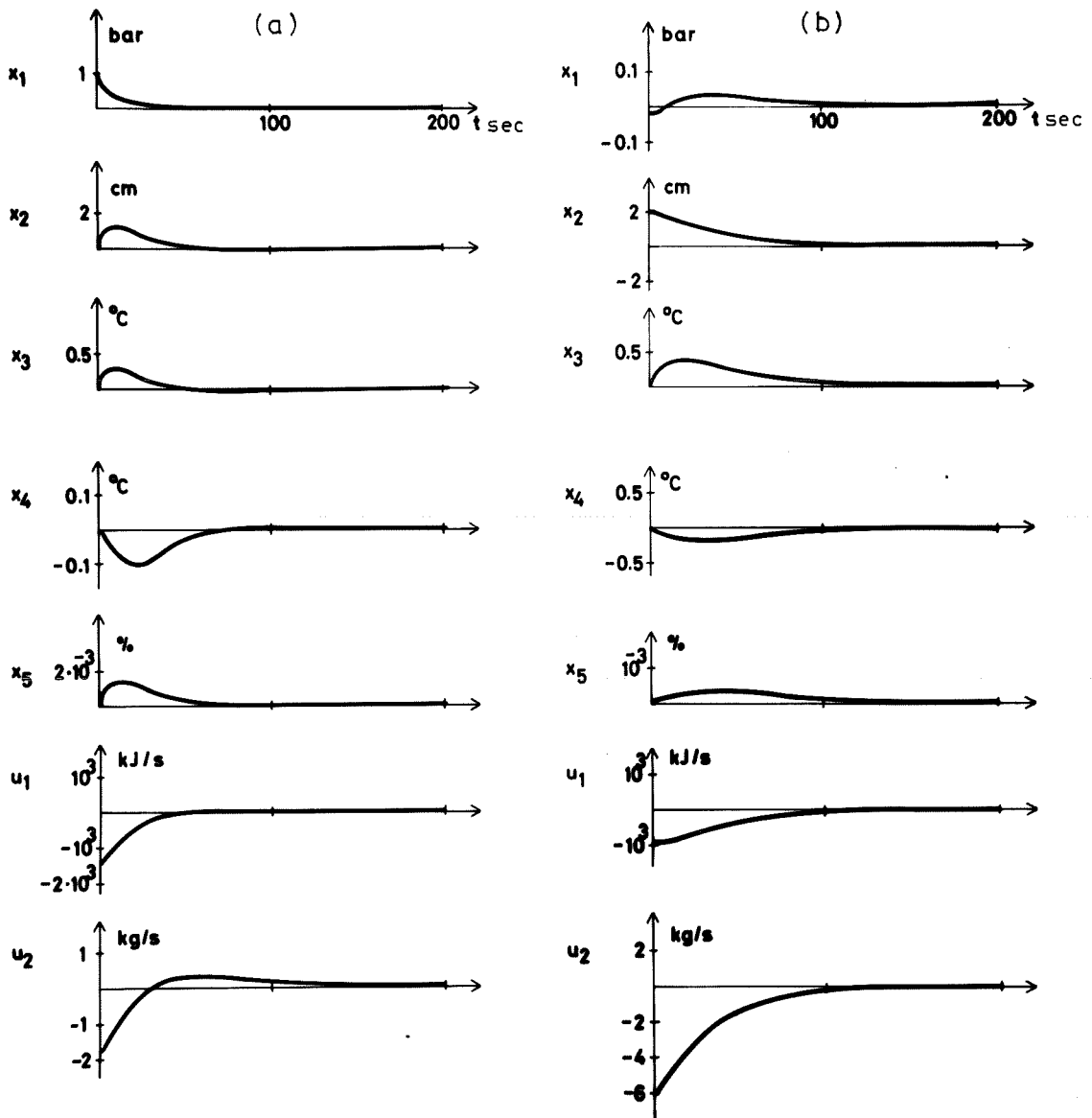


Fig. 6 - Responses of state variables to an initial value disturbance of (a)  $x_1(t)$  and (b)  $x_2(t)$ . Control law I is used.

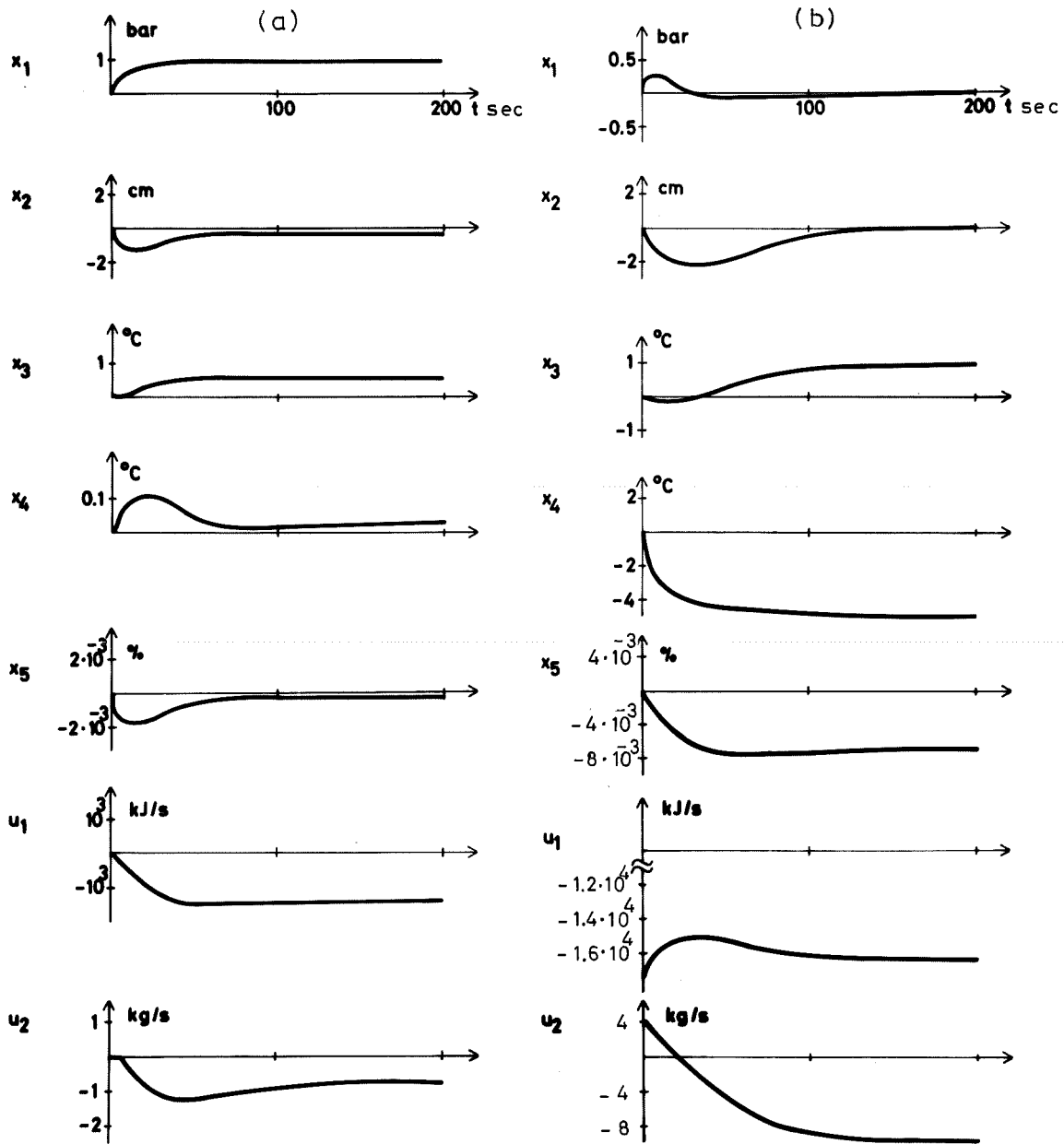


Fig. 7 - Responses of state variables to a 10% step change of the load. Control law I is used (a) without feedforward and (b) with feedforward.

To improve the response of  $x_2(t)$  the 22-element of  $Q_1$  was increased. After some iterations the final choice of the two non-zero elements of  $Q_1$  were

$$q_{11}' = 10^{-2}$$
$$q_{22}' = 10^4$$

The matrix  $Q_2$  was not altered. Fig. 8 and 9 show the responses to the disturbances using  $\alpha = 1$ . The corresponding control law will be called control law II. It is now appraised that the magnitude of  $u_2(t)$  should not be further increased. The large positive value of  $u_2(t)$  during the first moment after the load decrease is due to the increased drum pressure which causes a sudden decrease of the drum level, see Fig. 2c. The eigenvalues of the closed system matrix (A-BL) are in this case

$$-7.55 \cdot 10^{-2} \pm 5.12 \cdot 10^{-2} i$$
$$-1.41 \cdot 10^{-1} \pm 1.70 \cdot 10^{-2} i$$
$$-4.90 \cdot 10^{-2}$$

Numerical values of the used feedback and feedforward matrices both for the continuous and discrete cases are found in Appendix A. A detailed presentation of the programs used to compute the control law is given in {5}.



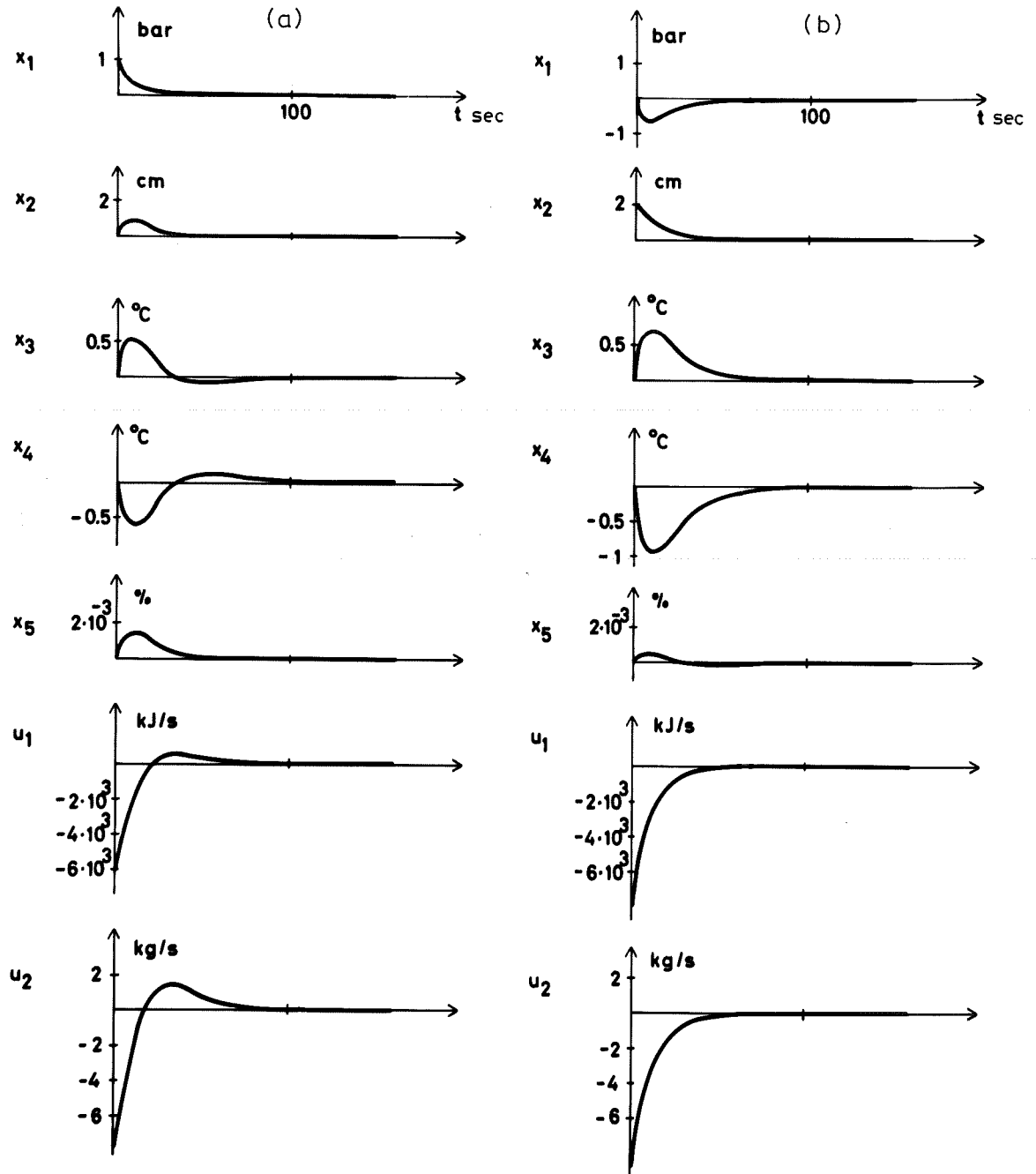


Fig. 8 - Responses of state variables to an initial value disturbance of (a)  $x_1(t)$  and (b)  $x_2(t)$ . Control law II is used.

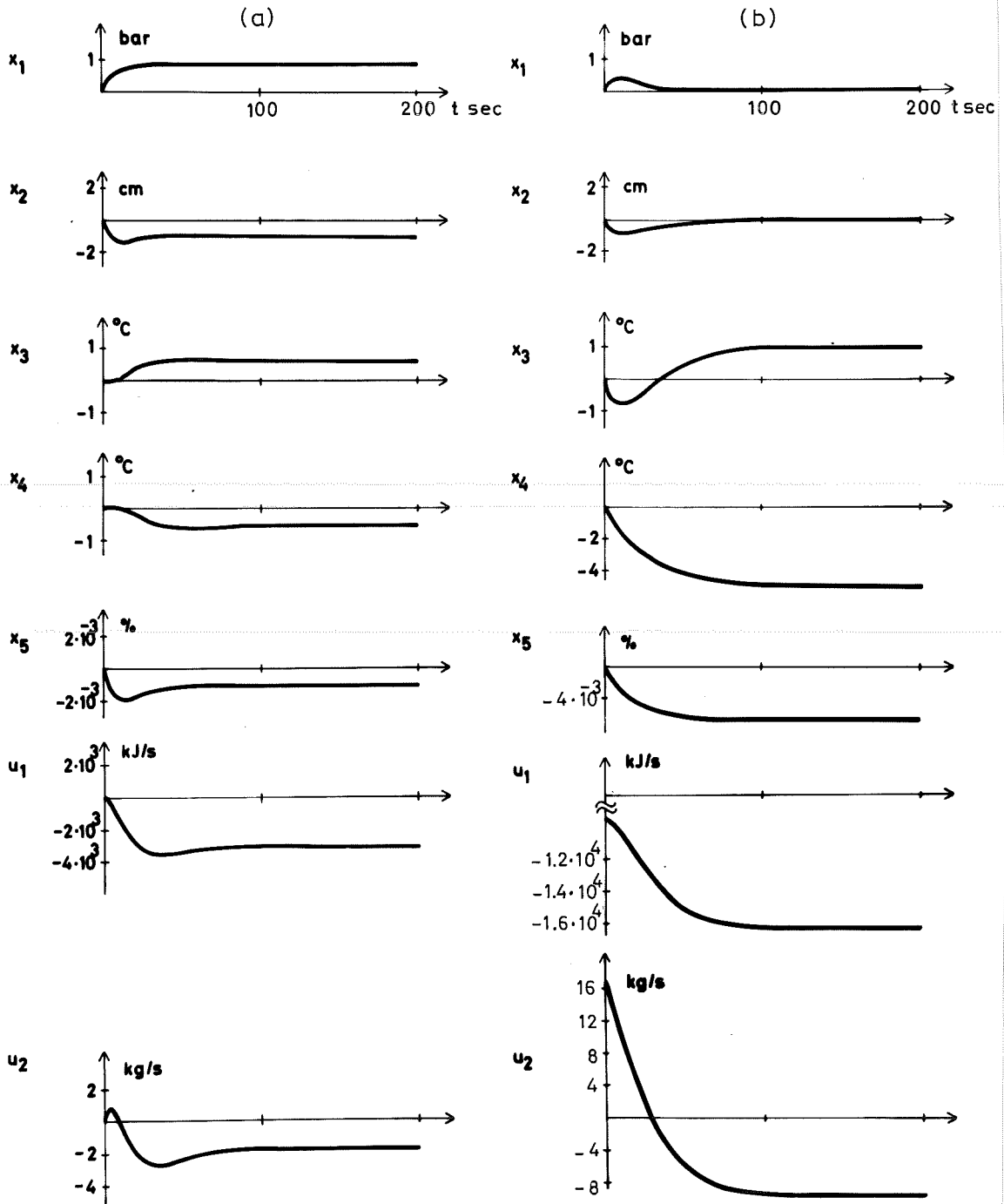


Fig. 9 - Responses of state variables to a 10% step change of the load. Control law II is used (a) without feedforward and (b) with feedforward.

### 7. CHOICE OF SAMPLING INTERVAL

The choice of the sampling interval is usually a difficult problem. A common method to find a suitable sampling interval is to determine the value of the loss functional for different sampling intervals. The value of the loss functional, which is a measure of the quality of control, will increase quadratically with increasing length of the sampling interval. There are methods available which give good estimates of the influence of the sampling interval on the loss functional, see e.g. [6]. However, in this study a very rough estimate is used.

The increase of the loss functional due to increasing sampling interval will be judged from the first two diagonal elements of the stationary S matrix. These elements correspond to the controlled variables  $x_1(t)$  and  $x_2(t)$ . In Table 1 numerical values for the case of control law II are given.

Sampling interval	$S_{11} \cdot 10^{-1}$	$S_{22} \cdot 10^{-4}$
0 sec	1.734897	1.450922
1	1.735418	1.451257
2	1.736984	1.452263
5	1.748047	1.459299
10	1.789885	1.484359

Table 1. The first two diagonal elements of the stationary S matrix for different sampling interval

In Fig. 10 the percentile increase  $\Delta S$  of the elements are plotted against the sampling interval.

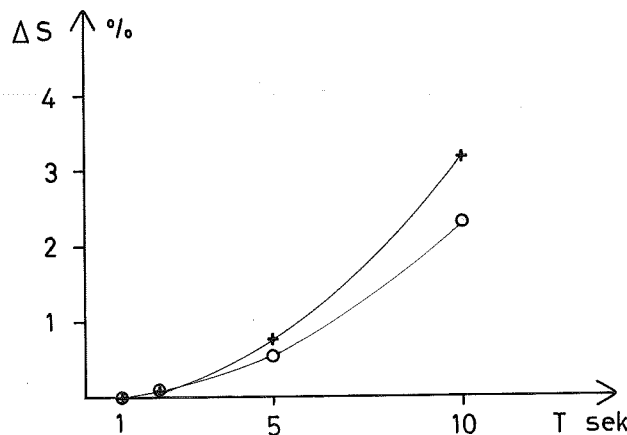


Fig. 10 - The increase  $\Delta S$  of the 11-element (x) and the 22-element (o) for different sampling intervals.

The two diagonal elements increase with approximately 3% for a sampling interval of 10 sec. The increase of the loss functional can roughly be estimated to the same amount. An increase less than 5% is acceptable and we choose a sampling interval of 10 sec.



The Kalman filter equation is given by

$$\hat{x}(t + 1) = \phi \hat{x}(t) + \Gamma u(t) + K[y(t) - \theta \hat{x}(t)] \quad (8.5)$$

If we assume that the covariance matrix  $R_2$  of the measurement noise equals zero the steady state covariance matrix of the reconstruction error and the filter gains can be computed in the following manner [7]. The control variables are omitted since as usual they only represent an additional term in the equations and do not influence the solution. Then equation (8.4) gives

$$x(t + 1) = \phi x(t) + \Gamma_e e_1(t) \quad (8.6 a)$$

$$y(t) = \theta x(t) \quad (8.6 b)$$

The input-output relation is given by

$$y(t) = \frac{B_2 z^{-2} + \dots + B_n z^{-n}}{1 + a_1 z^{-1} + \dots + a_n z^{-n}} e_1(t) \quad (8.7)$$

where  $a_1, \dots, a_n$  are the coefficients of the characteristic polynomial and  $B_2, \dots, B_n$  are coefficient matrices of order  $2 \times 1$ . The coefficient matrix  $B_1$  is zero since  $B_1$  equals  $\theta \cdot \Gamma_e$ .

Assume that the system is initialized at  $t = 0$ . Equation (8.6 a) then gives

$$x(t) = \phi^t x(0) + \sum_{s=0}^{t-1} \phi^{t-1-s} \Gamma_e e_1(s) \quad (8.8)$$

Given  $y(0), \dots, y(t)$ ,  $t > n$  the stochastic variables  $e(0), \dots, e(t-2)$  can be computed exactly using equation (8.7). Assume that the initial value  $x(0)$  is known then we can compute  $x(t-1)$  exactly from equation (8.8). Given  $x(t-1)$  the minimum mean square estimate of  $x(t+1)$  is

$$\hat{x}(t + 1) = \phi^2 x(t - 1) \quad (8.9)$$

The true value of state vector at  $t+1$  is

$$x(t + 1) = \phi^2 x(t - 1) + \phi \Gamma_e e_1(t - 1) + \Gamma_e e_1(t)$$

Then the steady state value of the covariance matrix  $P$  of the reconstruction error is

$$P = E\{x(t) - \hat{x}(t)\}\{x(t) - \hat{x}(t)\}^T = \phi \Gamma_e r_1 \Gamma_e^T \phi^T + \Gamma_e r_1 \Gamma_e^T \quad (8.10)$$

or

$$P = r_1 (\varphi_6 \varphi_6^T + \Gamma_e \Gamma_e^T) \quad (8.11)$$

where  $\varphi_6$  is the sixth column of  $\phi$ . To compute the filter gains we use equation (8.6) and the fact that  $\theta \Gamma_e$  is zero. We get

$$y(t) = \theta \phi^2 x(t-2) + \theta \phi \Gamma_e e_1(t-2) \quad (8.12)$$

$\theta \phi \Gamma_e$  is nonzero and we can solve equation (8.12) using the pseudo-inverse of  $\theta \phi \Gamma_e$ . Hence

$$e_1(t-2) = (\theta \phi \Gamma_e)^\dagger [y(t) - \theta \phi^2 x(t-2)] \quad (8.13)$$

Combining this equation and equation (8.6 a) we get the following recursive equation for the state vector

$$x(t-1) = \phi x(t-2) + (\theta \phi \Gamma_e)^\dagger [y(t) - \theta \phi^2 x(t-2)] \quad (8.14)$$

Equations (8.9) and (8.14) now give the filter equation (8.5) and the filter gains are

$$K = \phi^2 \Gamma_e (\theta \phi \Gamma_e)^\dagger \quad (8.15)$$

Notice that the filter gains are not uniquely determined. This fact can be exploited to adjust the weight given to the different measured variables in the Kalman filter. Equation (8.13) gives us two equations which both can be used to compute the scalar  $e_1(t-2)$ . The use of these equations can be weighted as

$$e_1(t-2) = \beta e_1'(t-2) + (1-\beta) e_1''(t-2)$$

where  $e_1'(t-2)$  and  $e_1''(t-2)$  are computed from the first and second equation of (8.13) respectively.  $\beta$  is the weighting factor. The eigenvalues of matrix  $(\phi - K\theta)$ , which give the dynamics of the reconstruction error will be independent of the factor  $\beta$ , only if these two equations are identical. In the boiler application there is a small difference between the two equations given by (8.13) which will slightly alter the eigenvalues when  $\beta$  is changed. Choosing  $\beta$  so that  $k_{11}$  roughly equals  $k_{22}$  we get

$$K = \begin{bmatrix} 0.69 & -28.7 \\ -0.01 & 0.44 \\ 0.21 & -8.72 \\ 0.21 & -8.94 \\ -0.001 & 0.06 \\ 0.88 & -36.9 \end{bmatrix} \quad (8.16)$$

The eigenvalues of  $\phi - K\theta$  are

- 0.00
- 0.00
- 0.41
- 0.32
- 0.99
- 0.83

There is one eigenvalue very near the unit circle in the complex plane. This means that the transient response of the reconstruction error will contain a very slow mode. But this also means that the filter equation is very sensitive to changes of the process parameters. In Appendix D the following expression is derived for the steady state reconstruction error

$$\tilde{x}_0 = (I - \phi + K\theta)^{-1}(\phi^* - \phi)x_0$$

where  $\phi^*$  is the disturbed process matrix.

We have

$$(I - \phi + K\theta)^{-1} = \begin{bmatrix} -5.5 \cdot 10^{-1} & 5.8 \cdot 10^3 & -2.2 \cdot 10^1 & -1.7 \cdot 10^1 & -4.8 \cdot 10^4 & 5.7 \cdot 10^{-1} \\ -1.3 \cdot 10^{-2} & 1.4 \cdot 10^2 & -5.2 \cdot 10^{-1} & -4.0 \cdot 10^{-1} & -1.1 \cdot 10^3 & -1.4 \cdot 10^{-2} \\ -5.7 \cdot 10^{-1} & 2.8 \cdot 10^3 & -8.8 & -8.0 & -2.3 \cdot 10^4 & 5.8 \cdot 10^{-2} \\ -6.1 \cdot 10^{-1} & 2.9 \cdot 10^3 & -1.1 \cdot 10^1 & -6.5 & -2.4 \cdot 10^4 & 1.1 \cdot 10^{-1} \\ 4.6 \cdot 10^{-3} & 2.0 & -4.8 \cdot 10^{-3} & -3.4 \cdot 10^{-3} & -1.4 \cdot 10^1 & -1.7 \cdot 10^{-3} \\ -2.2 & 5.7 \cdot 10^3 & -2.2 \cdot 10^1 & -1.7 \cdot 10^1 & -4.7 \cdot 10^4 & 2.0 \end{bmatrix}$$

The large elements of the second and fifth column of  $(I - \phi + K\theta)^{-1}$  indicate that the Kalman filter is very sensitive to changes of the parameters of the second and fifth row of  $\phi$ . If we change the 25:th element of  $\phi$  1% and let the steady state value of the state vector  $x_0$  correspond to a step change of  $v_1$  of 1 bar (a 10% load change) the steady state reconstruction error is

$$\tilde{x}_0 = \begin{bmatrix} 1.75 \\ 4.18 \cdot 10^{-2} \\ 8.37 \cdot 10^{-1} \\ 8.72 \cdot 10^{-1} \\ 5.86 \cdot 10^{-4} \\ 1.71 \end{bmatrix}$$



These reconstruction errors are not acceptable. Especially the two controlled variables  $x_1(t)$  and  $x_2(t)$  will deviate considerably from their steady state value. The stationary P matrix given by equation (8.11) equals

$$P = \begin{bmatrix} 5.4 \cdot 10^{-4} & -1.3 \cdot 10^{-5} & 5.5 \cdot 10^{-5} & 1.0 \cdot 10^{-4} & -1.7 \cdot 10^{-6} & 9.6 \cdot 10^{-4} \\ -1.3 \cdot 10^{-5} & 3.1 \cdot 10^{-7} & -1.3 \cdot 10^{-6} & -2.4 \cdot 10^{-6} & 4.0 \cdot 10^{-8} & -2.3 \cdot 10^{-5} \\ 5.5 \cdot 10^{-5} & -1.3 \cdot 10^{-6} & 5.7 \cdot 10^{-6} & 1.0 \cdot 10^{-5} & -1.7 \cdot 10^{-7} & 9.8 \cdot 10^{-5} \\ 1.0 \cdot 10^{-4} & -2.4 \cdot 10^{-6} & 1.0 \cdot 10^{-5} & 1.9 \cdot 10^{-5} & -3.1 \cdot 10^{-7} & 1.8 \cdot 10^{-4} \\ -1.7 \cdot 10^{-6} & 4.0 \cdot 10^{-8} & -1.7 \cdot 10^{-7} & -3.1 \cdot 10^{-7} & 5.1 \cdot 10^{-9} & -2.9 \cdot 10^{-6} \\ 9.6 \cdot 10^{-4} & -2.3 \cdot 10^{-5} & 9.8 \cdot 10^{-5} & 1.8 \cdot 10^{-4} & -2.9 \cdot 10^{-6} & 3.4 \cdot 10^{-3} \end{bmatrix}$$

The standard deviations  $\sigma_{x_i}^2$  of the reconstruction errors then are

$$\begin{aligned} \sigma_{x_1}^2 &= 2.3 \cdot 10^{-2} \text{ bar} \\ \sigma_{x_2}^2 &= 5.6 \cdot 10^{-4} \text{ m} \\ \sigma_{x_3}^2 &= 2.4 \cdot 10^{-3} \text{ }^\circ\text{C} \\ \sigma_{x_4}^2 &= 4.3 \cdot 10^{-3} \text{ }^\circ\text{C} \\ \sigma_{x_5}^2 &= 7.1 \cdot 10^{-5} \% \\ \sigma_{x_6}^2 &= 5.8 \cdot 10^{-2} \text{ bar} \end{aligned} \tag{8.17}$$

The values for the 3:rd, 4:th and 5:th components of the state vector are unrealistically small since obviously the model is not that accurate.

One way to introduce uncertainties in the model is to add white noise with a given variance to each component of the state vector. Choosing standard deviations of this noise as roughly 1% of the maximum deviations of the state variables when the load is changed 10% we get

$$R_1 = \begin{bmatrix} 10^{-4} & 0 & 0 & 0 & 0 & 0 \\ 0 & 4 \cdot 10^{-8} & 0 & 0 & 0 & 0 \\ 0 & 0 & 10^{-4} & 0 & 0 & 0 \\ 0 & 0 & 0 & 2.5 \cdot 10^{-3} & 0 & 0 \\ 0 & 0 & 0 & 0 & 10^{-8} & 0 \\ 0 & 0 & 0 & 0 & 0 & 1.69 \cdot 10^{-3} \end{bmatrix} \tag{8.18}$$

The covariance matrix of the measurement noise is chosen to

$$R_2 = \begin{bmatrix} 10^{-4} & 0 \\ 0 & 10^{-7} \end{bmatrix} \quad (8.19)$$

Notice that the nonzero elements of  $R_2$  have roughly the same magnitude as the variance of the reconstruction errors of  $x_1(t)$  and  $x_2(t)$  respectively.

Numerical values of the obtained filter gains are given in Appendix A. The eigenvalues of  $\phi - K\theta$  are

$$0.10 \pm 0.10 i$$

$$0.26 \pm 0.14 i$$

$$0.51$$

$$0.34$$

and the sensitivity matrix is

$$(I - \phi + K\theta)^{-1} = \begin{bmatrix} -7.1 \cdot 10^{-4} & 7.6 & -2.8 \cdot 10^{-2} & -2.2 \cdot 10^{-2} & -6.2 \cdot 10^1 & 7.4 \cdot 10^{-1} \\ -8.0 \cdot 10^{-5} & 8.5 \cdot 10^{-1} & -3.1 \cdot 10^{-3} & -2.4 \cdot 10^{-3} & -6.9 & -1.3 \cdot 10^{-2} \\ -3.1 \cdot 10^{-1} & -1.2 & 1.6 & 5.9 \cdot 10^{-3} & -3.9 \cdot 10^1 & 1.1 \cdot 10^{-1} \\ -3.3 \cdot 10^{-1} & -1.2 & 1.9 \cdot 10^{-2} & 1.8 & 9.8 & 1.1 \cdot 10^{-1} \\ 4.8 \cdot 10^{-3} & -1.3 \cdot 10^{-2} & 2.5 \cdot 10^{-3} & 2.3 \cdot 10^{-3} & 1.9 & -2.5 \cdot 10^{-3} \\ -1.6 & 1.4 \cdot 10^1 & -4.2 \cdot 10^{-1} & -3.3 \cdot 10^{-1} & -9.5 \cdot 10^1 & 2.3 \end{bmatrix}$$

Notice that no eigenvalue is close to the unit circle in the complex plane and that the elements of the sensitivity matrix have been reduced with about a factor 100. The standard deviations  $\sigma_{x_i}^v$  of the reconstruction errors in this case are

$$\sigma_{x_1}^v = 3.0 \cdot 10^{-2} \text{ bar}$$

$$\sigma_{x_2}^v = 8.5 \cdot 10^{-4} \text{ m}$$

$$\sigma_{x_3}^v = 1.2 \cdot 10^{-2} \text{ }^\circ\text{C}$$

$$\sigma_{x_4}^v = 5.6 \cdot 10^{-2} \text{ }^\circ\text{C}$$

$$\sigma_{x_5}^v = 1.5 \cdot 10^{-4} \%$$

$$\sigma_{x_6}^v = 6.2 \cdot 10^{-2} \text{ bar}$$

Compared to (8.17) the standard deviations of the 3:rd, 4:th and 5:th components of the state vector have increased about ten times. Thus the filter gains obtained using the disturbed model give reasonable properties of the filter equation and these gains will be used.

### 9. IMPLEMENTATION OF CONTROL LAW ON A PROCESS CONTROL COMPUTER

The whole problem was simulated on the hybrid computer at the Research Institute of National Defense in Stockholm, Sweden. The process was patched on the analog computer EAI 8800 and the control law was implemented on the digital computer EAI 640. The details of the simulation are found in {3}.

A simplified flow diagram of the control algorithm is shown in Fig. 11.

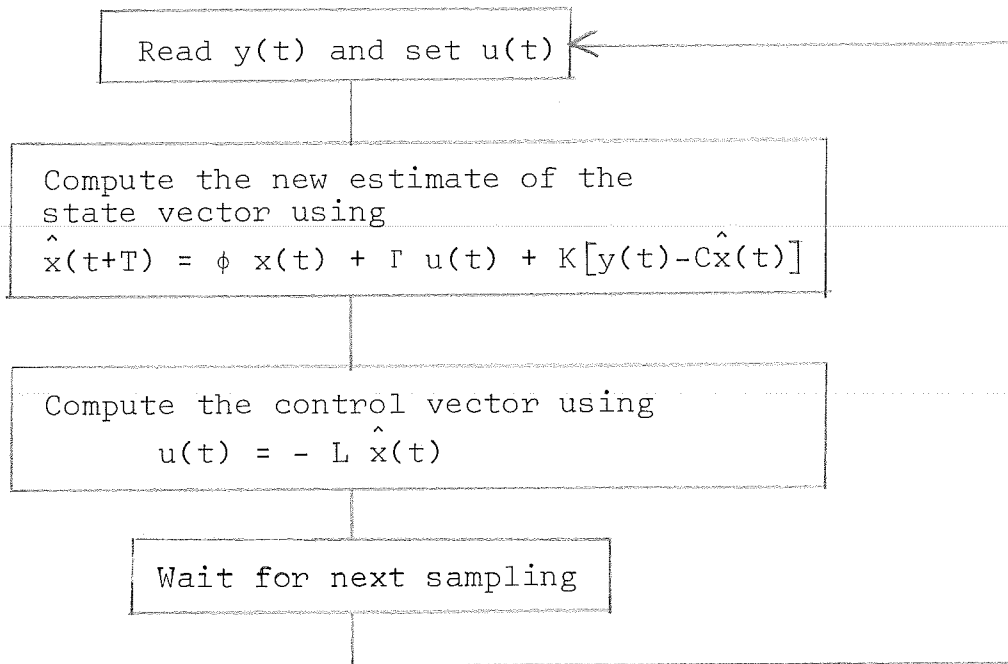


Fig. 11 - Simplified flow diagram of control algorithm

The matrices  $\phi$  and  $\Gamma$  are the sampled A and B matrices. Notice that the filter equation is of the 6:th order since one state has been added for the load disturbance. The filter equation gives an estimate of the state vector and the load disturbance one sampling interval ahead and the control vector can be computed using this estimate. When the next sampling interrupt occurs the control variables are set and a new measurement of the output variables is made. Numerical figures of sampled matrices are given in Appendix A.

The control law was implemented using fix point arithmetic and single precision. The word length of the computer is 16 bits including the sign bit which gives an accuracy of about 4

decimal digits. The numbers in the computer are regarded as fractionals. Then we must make sure that no constants or sums become larger than one. Otherwise overflow will occur. Each estimated state variable is scaled according to the largest matrix element on the right hand side in the filter equation. These scale factors are then introduced in the coefficients of the L matrix. The control variables are then also scaled according to the largest element. Before the storing and setting of  $\hat{x}(t)$  and  $u(t)$  they are rescaled. It is obvious that some caution must be exercised so that the accuracy not unnecessarily is decreased and that the scaling requires a considerable knowledge about intermediate results during the calculations.

The control algorithm (CALG) is programmed in assembler language. The program listings are given in Appendix C. The matrix calculations are performed using subroutines for vector addition (VADD) vector subtraction (VSUB) and matrix-vector multiplication (MVMULT) The rescaling subroutine is called RESCA. A detailed presentation is found in {8}. The core memory requirements for the control algorithm and subroutines are shown in Table 3. Figures are given for a 6:th and a 15:th order system both with 2 inputs and 2 outputs.

n	6	15
CALG	121 words	157 words
VADD	49	49
VSUB	5	5
MVMULT	81	81
RESCA	45	45
Matrix storage array	84	345
SUM	385	682

Table 3. The storage requirements for the control algorithm and subroutines for a 6:th and a 15:th order system both with 2 inputs and 2 outputs.

The program list for CALG apply to a 15:th order system with 10 inputs and 10 outputs. There is some unnecessary storage arrays in CALG since some intermediate results are saved.

The execution time for CALG is 6.7 ms.

## 10. SIMULATION

In the analog simulations of the boiler control it was assumed that all state variables and the disturbance  $v_1$  could be measured directly. Hence the Kalman filter was not included.

Fig. 12 gives the open loop responses of the state variables when the disturbance  $v_1$  is a stochastic process given by equation (4.7). Fig. 13 and 14 give the responses of the state variables when control law II, without and with feedforward respectively, is used. Notice that the realizations of  $v_1(t)$  are different in the figures referred to above. A measurement of the variance of  $x_1(t)$  and  $x_2(t)$  on the analog computer gave

$$\begin{aligned} E x_1^2(t) &= 1.1 \cdot 10^{-3} \text{ bar}^2 \\ E x_2^2(t) &= 6.0 \cdot 10^{-4} \text{ cm}^2 \end{aligned} \quad (10.1)$$

The results of the hybrid simulations are presented in Fig. 15, 16, 17, 18.

Fig. 15a gives the responses of the state variables to a step change of the load disturbance  $v_1$ . The corresponding estimated state and disturbance variables are given in Fig. 15b. Control law II with feedforward is used and the filter gains correspond to the covariance matrices  $R_1$  and  $R_2$  given by equations (8.18) and (8.19). Notice that the control variables are zero during the first two sampling intervals. This follows from that the control algorithm CALG is given starting values for  $\hat{x}(0)$  and  $u(0)$  which equal zero.

Fig. 16 illustrates the sensitivity of the Kalman filter using the filter gains given by equation (8.16). The model disturbance is a change of the 25:th element of the matrix A with 0.25%. The estimation errors of especially  $x_1(t)$ ,  $x_2(t)$  and  $v_1(t)$  are considerable.

Fig. 17 which should be compared with Fig. 14 gives the responses of the state variables to the stochastic process  $v_1(t)$  defined by equation (4.7). The estimates are presented in Fig. 18. The measured variances of the two controlled variables are

$$\begin{aligned} E x_1^2(t) &= 2.8 \cdot 10^{-3} \text{ bar}^2 \\ E x_2^2(t) &= 6.6 \cdot 10^{-3} \text{ cm}^2 \end{aligned} \quad (10.2)$$

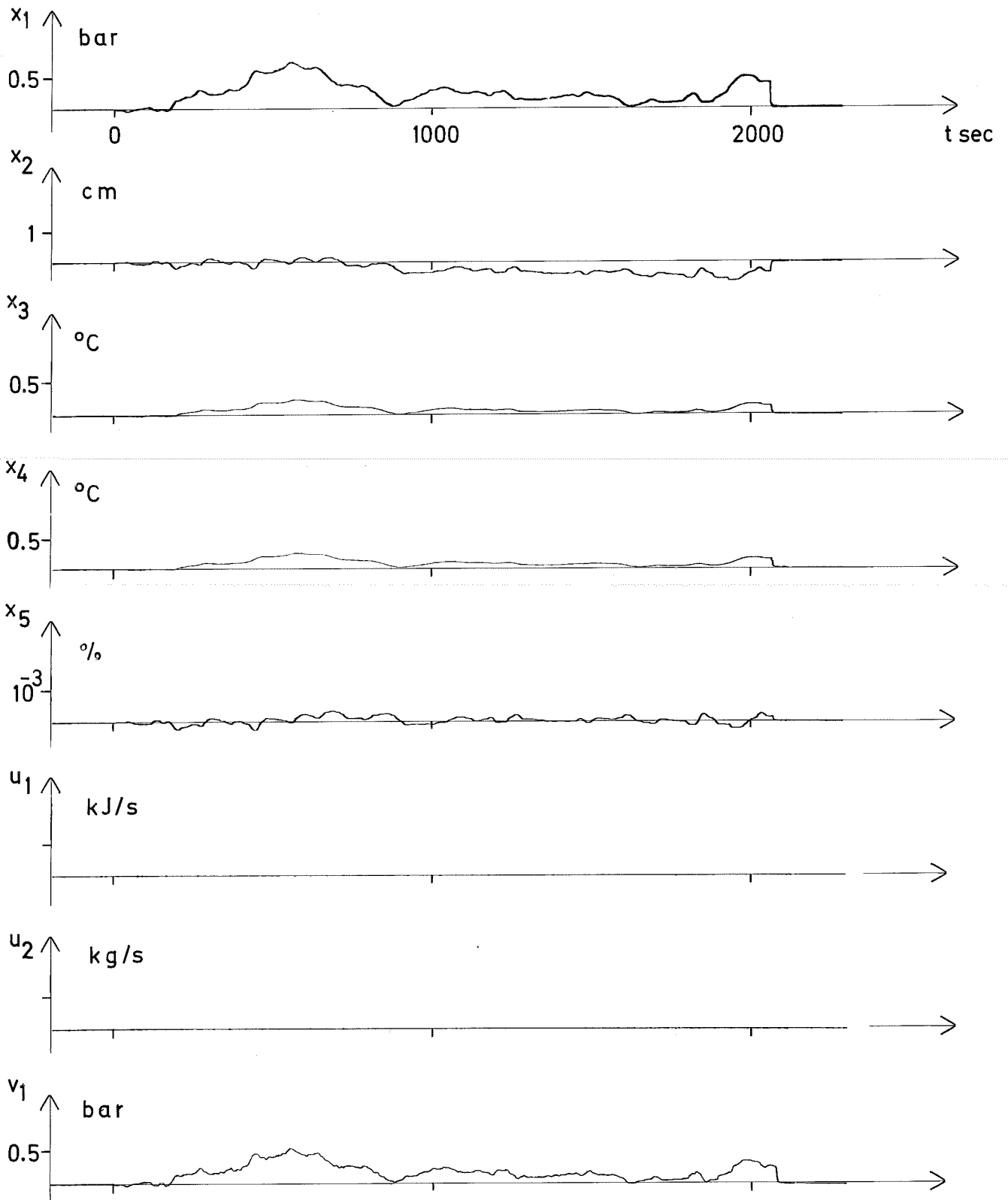


Fig. 12 - Open loop responses of the state variables. The disturbance  $v_1(t)$  is given by equation (4.7).

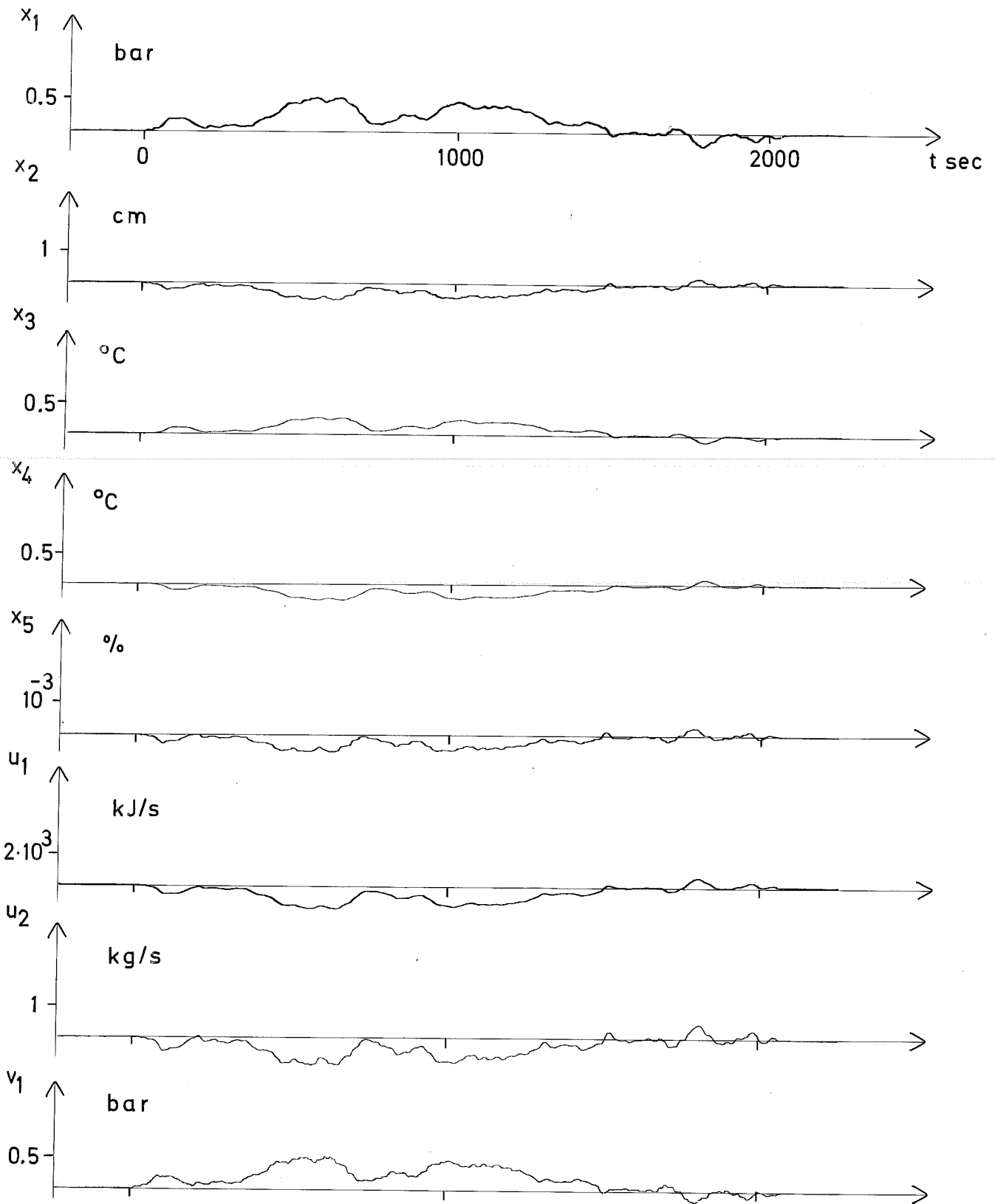


Fig. 13 - Responses of state variables to load disturbance  $v_1(t)$  defined by equation (4.7). Control law II without feedforward is used.

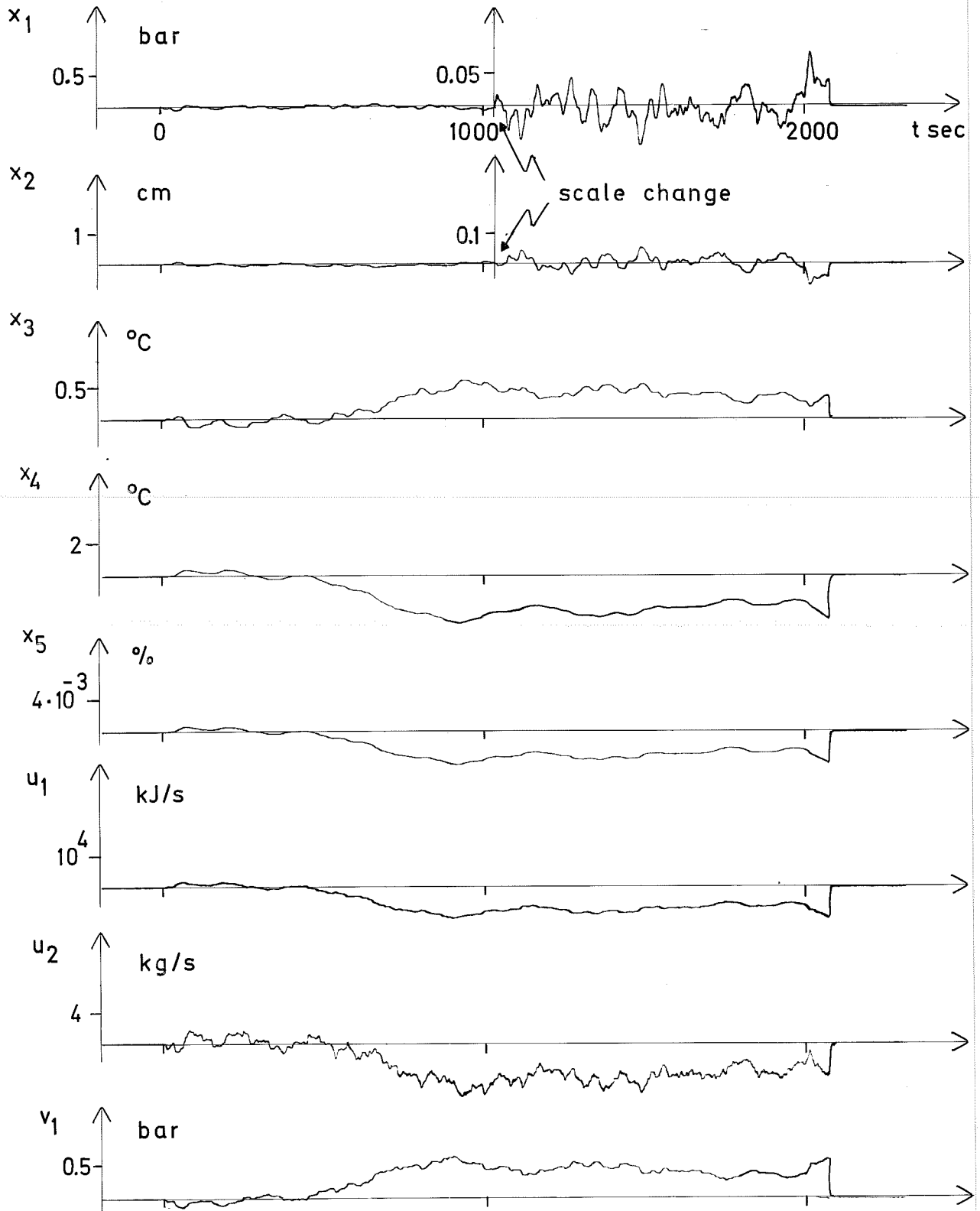


Fig. 14 - Responses of state variables to load disturbance  $v_1(t)$  defined by equation (4.7). Control law II with feedforward is used.



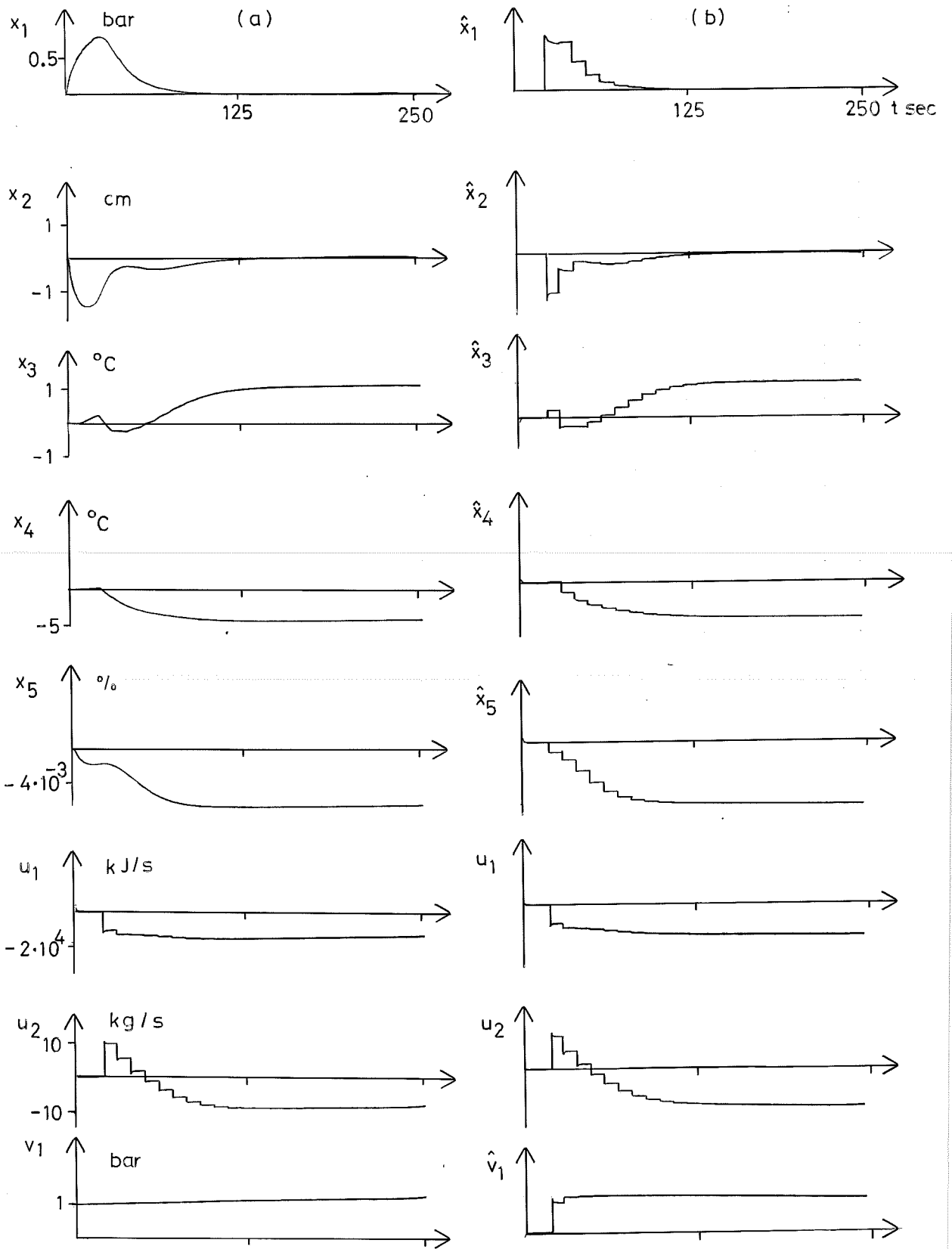


Fig. 15 - Responses of state variables (a) and estimated state and disturbance variables (b) to a step change of 1 bar of load disturbance  $v_1(t)$ . Control law II with feedforward is used. Covariance matrices  $R_1$  and  $R_2$  given by equations (8.18) and (8.19) define the filter gains used.

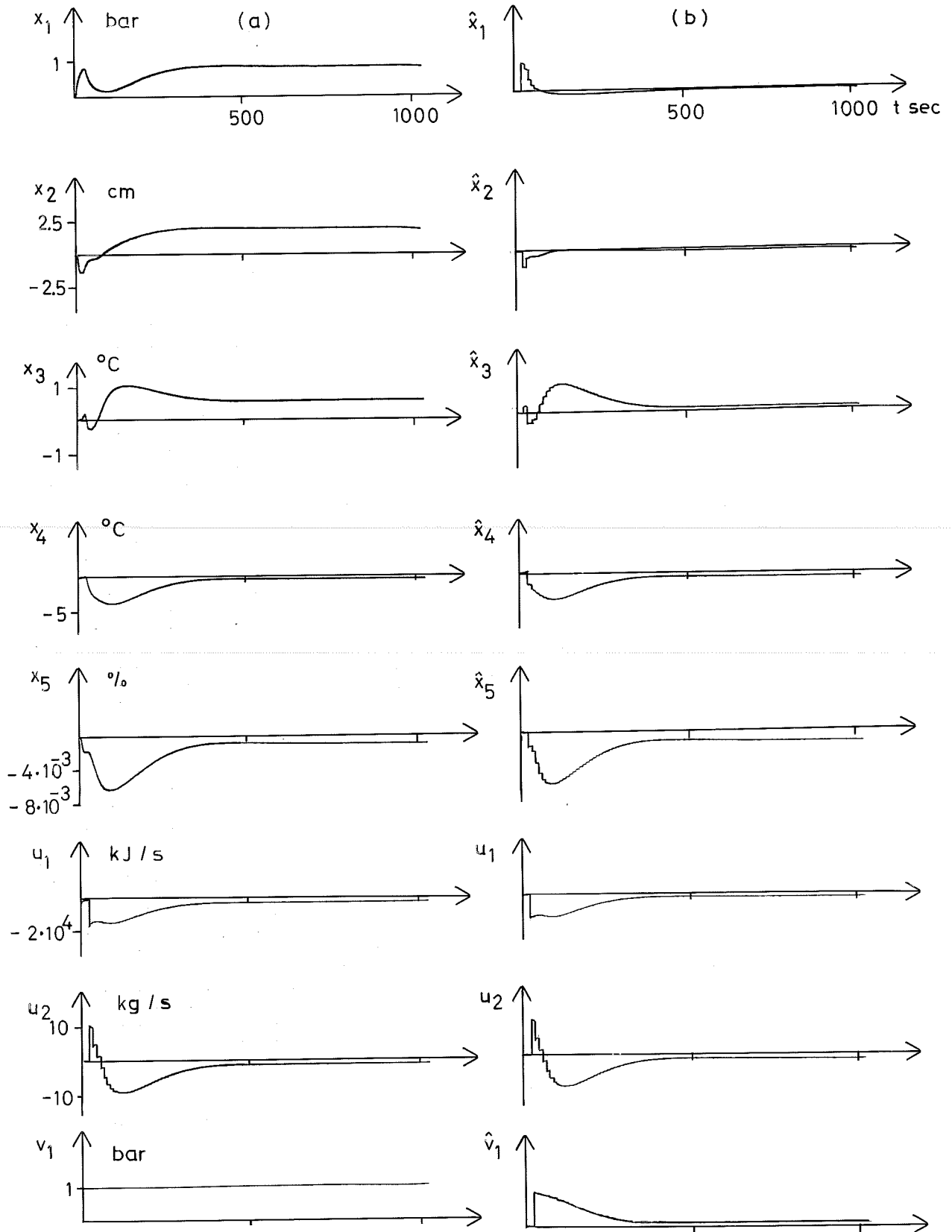


Fig. 16 - Responses of state variables (a) and estimated state and disturbance variables (b) to a step change of 1 bar of load disturbance  $v_1(t)$ . Control law II with feed-forward is used. The used filter gains are given by equation (8.16).

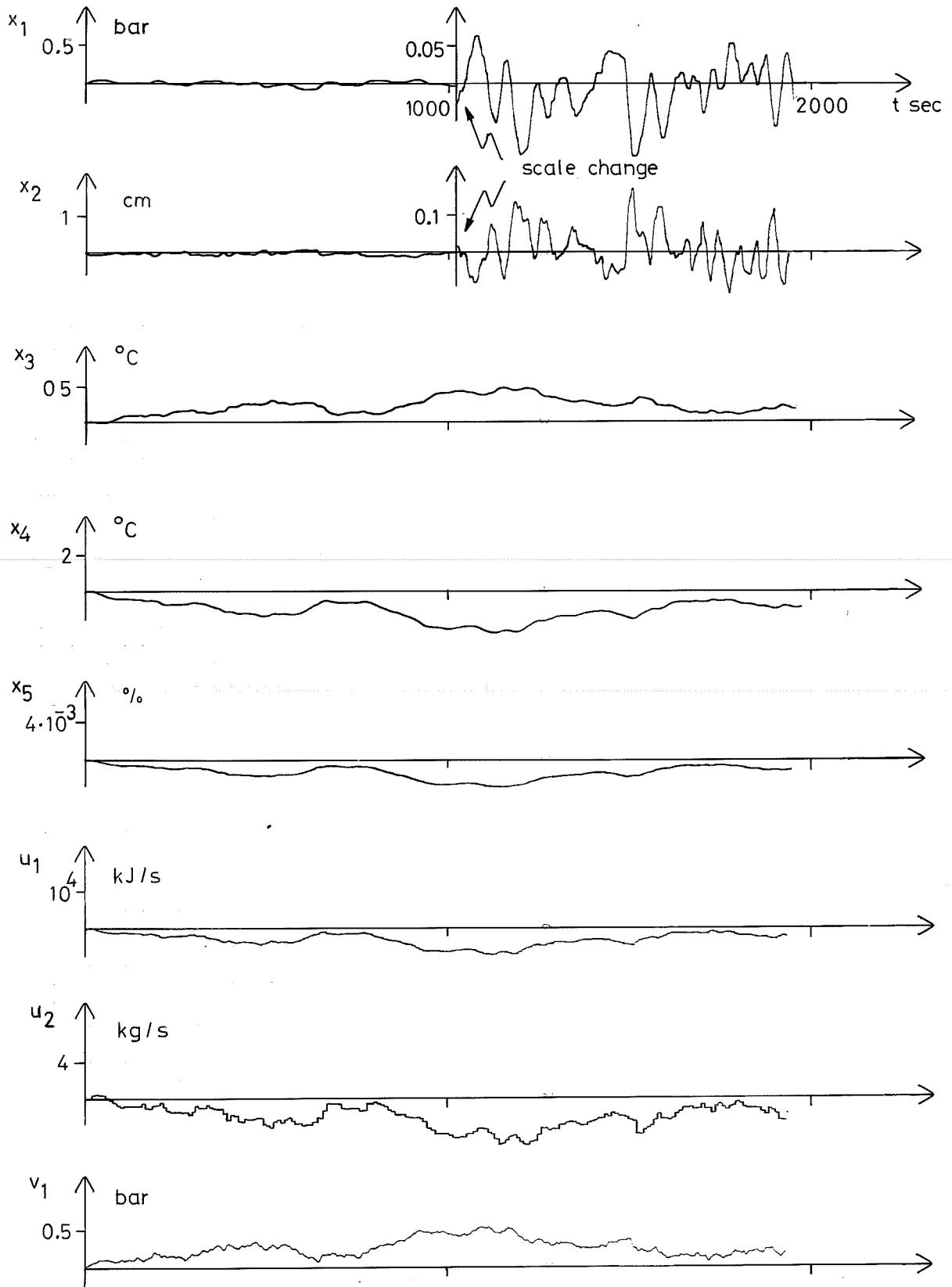


Fig. 17 - Responses of state variables to load disturbance  $v_1(t)$  given by equation (4.7). Control law II with feedforward is used. Covariance matrices  $R_1$  and  $R_2$  given by equations (8.18) and (8.19) define the filter gains used.

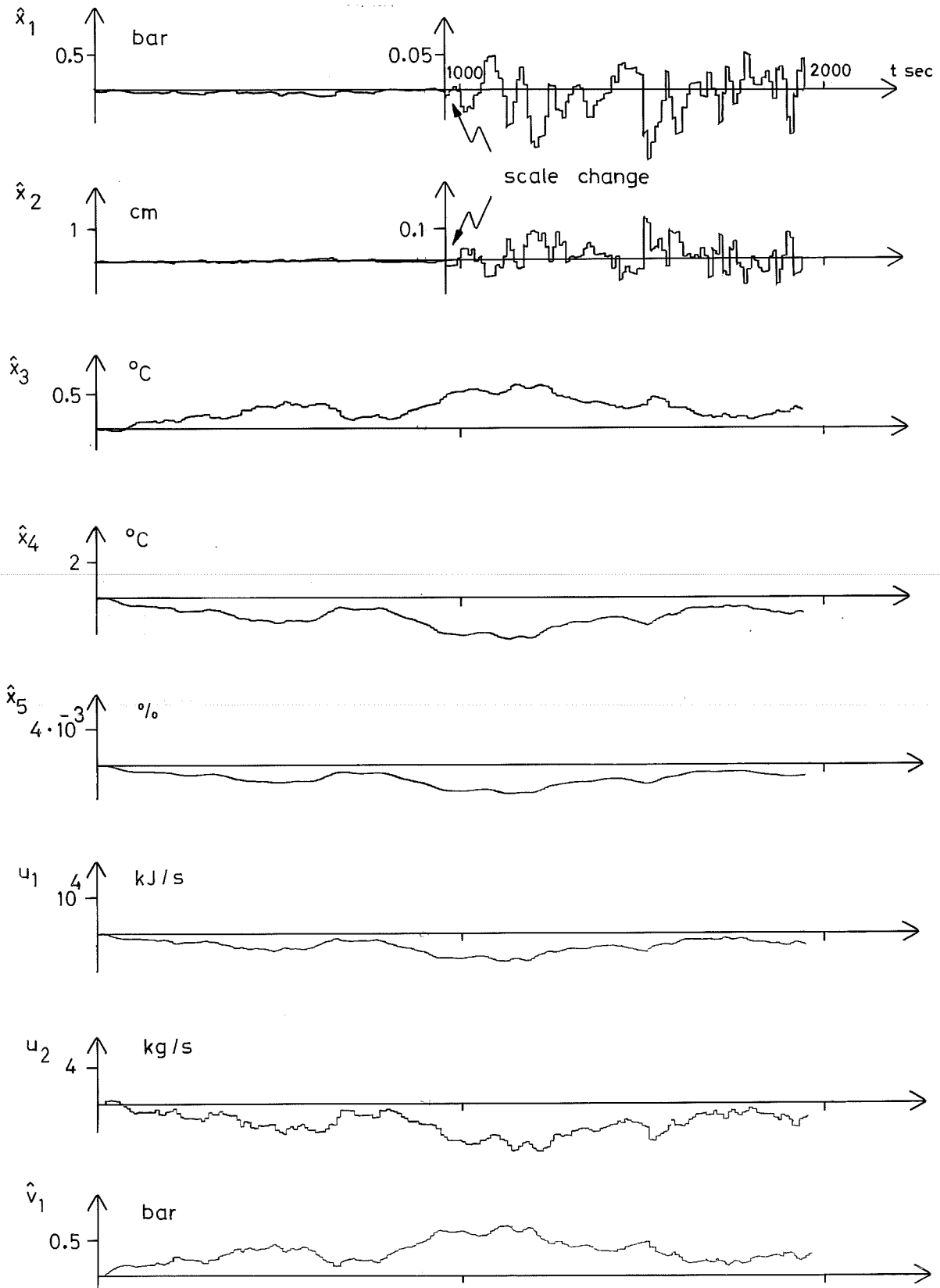


Fig. 18 - Responses of estimated state and disturbance variables to load disturbance  $v_1(t)$  given by equation (4.7). Control law II with feedforward is used. Covariance matrices  $R_1$  and  $R_2$  given by equations (8.18) and (8.19) define the filter gains used.

Compared to (10.1) the variances have increased roughly by a factor 2 and 10 respectively. The larger increase of the variance of  $x_2(t)$  was expected considering the frequency content of the control signals  $u_1(t)$  and  $u_2(t)$  in the continuous case, see Fig. 14.

#### 11. ACKNOWLEDGEMENTS

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## APPENDIX A

Numerical values of the matrices A, B, F and C of the boiler model used in this report are given. The values apply to a power station boiler with a maximum steam flow of about 350 t/h. The operating point is 90% of full load. Also the discrete boiler model matrices as well as the feedback, feedforward and filter matrices are given.

The continuous system matrices are:

MATRIX A

-1.2917081500-001  
 3.2907472400-003  
 7.1772161300-002  
 4.1127161600-002  
 3.6122785500-004

3.9622910300-002  
 -7.7913590600-005  
 -1.0041304100-001  
 -0.0000000000+000  
 3.5023976300-005

2.5023257900-002  
 1.2162226000-004  
 8.8726400000-004  
 -8.2254323200-002  
 4.25559545699-005

1.9119354000-002  
 -6.2135344500-001  
 -3.8507722100+000  
 -0.0000000000+000  
 -7.4327812999-002

MATRIX B

-0.0000000000+000  
 -0.0000000000+000  
 -0.0000000000+000  
 2.4884535800-005  
 -0.0000000000+000

1.3935922200-003  
 3.5944889200-005  
 -9.8919025499-003  
 -0.0000000000+000  
 -5.3430336699-006

MATRIX F

9.9473651899-002  
 -3.1814050400-003  
 -2.3209124400-002  
 -6.0000000000+000  
 -3.8138205800-004

MATRIX C

1 0 0 0 0 0  
 0 1 0 0 0 0

The discrete system matrices for  $T = 10$  sec:

MATRIX  $\phi$

3.3340778199-001	0.0000000000+000	1.3393072381-001	9.3340709882-002	2.7947667816+000
1.3097281272-002	1.0000000000+000	1.3841302424-003	1.7094978276-003	4.4004041692+000
2.1665335495-001	0.0000000000+000	4.0911537056-001	3.0770091572-002	1.6702744064+001
1.5301957496-001	0.0000000000+000	2.9803090464-002	4.5939395088-001	3.9128795615-001
1.4803653130-003	0.0000000000+000	4.2575896441-004	3.8294082174-004	4.6900439390-001

MATRIX  $\Gamma$

1.6417868651-005	-1.4696276466-003
2.1800773794-007	5.6284718163-004
3.6464543804-006	-6.1419331881-002
1.7175937227-004	2.3135682378-004
5.1649590092-008	-4.5215376170-005

SAMPLED MATRIX F

5.0700610003-001
-1.3561195318-002
5.7902031018-002
1.0504507399-001
-1.7297831589-003



CONTINUOUS FEEDBACK MATRIX L OF CONTROL LAW I

1.4851558112+003	4.7087735868+004	4.9025667321+002	4.4297695742+002	-1.0435741927+005
1.8410253907+000	3.1270232798+002	-1.8653363861-001	-2.1287153831-002	-2.1008260647+003

CONTINUOUS FEEDFORWARD MATRIX R OF CONTROL LAW I

1.7367719797+004
-4.1942240773+000

CONTINUOUS FEEDBACK MATRIX L OF CONTROL LAW II

6.6780720481+003	4.1803405923+005	1.3552544432+003	1.3718685270+003	-1.7532529445+006
8.0344481077+000	9.0843133789+002	4.8581215785-001	8.1554038939-001	-4.3102102217+003

CONTINUOUS FEEDFORWARD MATRIX R OF CONTROL LAW II

9.6400244546+003
-1.6681709147+001

SAMPLED FEEDBACK MATRIX L OF CONTROL LAW I

1.2585005228+003    4.52732256515+004    4.4963971804+002  
1.5363789856+000    2.7413239833+002    -2.0105521722-001

4.0898954185+002    -1.9188271739+005  
-5.0266359095-002    -1.9753315105+003

SAMPLED FEEDFORWARD MATRIX R OF CONTROL LAW I

1.6647427054+004  
-3.4646600566+000

SAMPLED FEEDBACK MATRIX L OF CONTROL LAW II

4.7553637319+003    2.9068339261+005    1.1620722849+003  
6.0422080146+000    6.4567754380+002    3.9813452707-001

1.1433438861+003    -1.6021789753+006  
6.4490091534-001    -3.8272280010+003

SAMPLED FEEDFORWARD MATRIX R OF CONTROL LAW II

1.0016191581+004  
-1.3177689952+001

Using  $R_1$  and  $R_2$  according to (8.18) and (8.19) the filter gain matrix  $K$  is

1.0073571793+000	-5.7569950558+000
6.3120600146-003	1.0757017094+000
3.7910573573-001	5.3174449364-001
4.0181264417-001	2.6182033238-001
-1.7632880588-003	7.1557604587-003
1.1591905537+000	-1.0347879562+001

APPENDIX B

Numerically it has been shown for several specific problems that the feedforward matrix R discussed in section 5 also can be computed in the following way. Consider the system

$$\frac{dx(t)}{dt} = Ax(t) + Bu(t) + Fv(t) \quad (1)$$

Set

$$\begin{aligned} v_k &= x_{n+k} & k &= 1, \dots, s \\ \dot{x}_{n+k} &= 0 & k &= 1, \dots, s \end{aligned}$$

and add these new state variables to the system equation (1).

We get

$$\frac{dx(t)}{dt} = \begin{bmatrix} A & F \\ 0 & 0 \end{bmatrix} x(t) + \begin{bmatrix} B \\ 0 \end{bmatrix} u(t) = A_1 x(t) + B_1 u(t) \quad (2)$$

Introduce the notation

$$\bar{A}_1 = [0 \ 0 \ \dots \ 0 \ a_{i+1}^1 \ \dots \ a_n^1]$$

where  $a_k^1$  is the k:th column of A. We require that the steady state error of the first i components of the state vector equals zero, and that i equals the number of control variables. The loss functional

$$V = \frac{1}{2} x^T(t_1) Q_0 x(t_1) + \frac{1}{2} \int_{t_0}^{t_1} \{ x^T(s) Q_1 x(s) + (\bar{A}_1 x(s) + B_1 u(s))^T Q_2 (\bar{A}_1 x(s) + B_1 u(s)) \} ds$$

then gives the control law

$$u(t) = -L_1(t) \begin{bmatrix} x_1(t) \\ \vdots \\ x_n(t) \end{bmatrix} - L_2(t) \begin{bmatrix} x_{n+1} \\ \vdots \\ x_{n+s} \end{bmatrix}$$

The stationary value of  $L_2(t)$  obtained when  $t_1 \rightarrow \infty$  is given by

$$L_2 = R$$

where R satisfy equation (5.7) using the stationary value of  $L_1(t)$ . Notice that  $\bar{A}_1 x_0 + B_1 u_0$  equals the left hand side of equation (5.4).

```

00001:          * SUBROUTINE  CALG
00002:          * COMMON AREA MATR
00003:          COMMON  MATR
00004:          00001  N    BSS    1
00005:          00001  M    BSS    1
00006:          00001  P    BSS    1
00007:          00341  FI   BSS   225
00008:          00226  GAMMA BSS   150
00009:          00226  C    BSS   150
00010:          00226  K    BSS   150
00011:          00226  L    BSS   150
00012:          * COMMON AREA VECT
00013:          COMMON  VECT
00014:          00017  XHAT  BSS    15
00015:          00012  U    BSS    10
00016:          00012  Y    BSS    10
00017:          * COMMON AREA ERROR
00018:          COMMON  ERROR
00019:          00010  EFDC  BSS     8
00020:          00012  E    BSS    10
00021:          00014  UTFY2 BSS    12          FILL UP TO 30
00022:          * EXT ASSEMBLER CALLING SEQUENCE
00023:          *          CALL CALG
00024:          * SUBROUTINES CALLED
00025:          *          VADD
00026:          *          VSUB
00027:          *          MVMULT
00028:  01000          00000          REL          0
00029:          NAME          CALG
00030:  00000  00000  00000  CALG  ADR          0
00031:  00001  60XXX          CALL          MVMULT,N,N,FI,XHAT,FX,E,0
00031:  00002  XXXXX 00000 ---
00031:  00003  XXXXX 00000 ---
00031:  00004  XXXXX 00003 ---
00031:  00005  XXXXX 00000 ---
00031:  00006  XXXXX 00122 ---
00031:  00007  XXXXX 00010 ---
00031:  00010  00000 00000 ---
00032:  00011  60XXX          F1          CALL          MVMULT,N,M,GAMMA,U,GU,E+1,
00032:  00012  XXXXX 00000 ---
00032:  00013  XXXXX 00001 ---
00032:  00014  XXXXX 00344 ---
00032:  00015  XXXXX 00017 ---
00032:  00016  XXXXX 00141 ---
00032:  00017  XXXXX 00011 ---
00032:  00020  00000 00000 ---
00033:  00021  60XXX          F2          CALL          VADD,N,FX,GU,FXPGU,E+2,0
00033:  00022  XXXXX 00000 ---
00033:  00023  XXXXX 00122 ---
00033:  00024  XXXXX 00141 ---
00033:  00025  XXXXX 00160 ---
00033:  00026  XXXXX 00012 ---
00033:  00027  00000 00000 ---
00034:  00030  60XXX          F3          CALL          MVMULT,P,N,C,XHAT,CX,E+3,0
00034:  00031  XXXXX 00002 ---
00034:  00032  XXXXX 00000 ---
00034:  00033  XXXXX 00572 ---
00034:  00034  XXXXX 00000 ---
00034:  00035  XXXXX 00177 ---
00034:  00036  XXXXX 00013 ---
00034:  00037  00000 00000 ---
00035:  00040  60XXX          F4          CALL          VSUB,P,Y,CX,YMCX,E+4,0
00035:  00041  XXXXX 00002 ---

```

APPENDIX C  
(continued)

00035:	00042	XXXXX	00031	----			
00035:	00043	XXXXX	00177	----			
00035:	00044	XXXXX	00211	----			
00035:	00045	XXXXX	00014	----			
00035:	00046	00000	00000	----			
00036:	00047	60XXX			F5	CALL	MVMULT,N,P,K,YMCX,KYCX,E+5
00036:	00050	XXXXX	00000	----			
00036:	00051	XXXXX	00002	----			
00036:	00052	XXXXX	01020	----			
00036:	00053	XXXXX	00211	----			
00036:	00054	XXXXX	00223	----			
00036:	00055	XXXXX	00015	----			
00036:	00056	00000	00000	----			
00037:	00057	60XXX			F6	CALL	VADD,N,FXPGU,KYCX,XHAT,E+
00037:	00060	XXXXX	00000	----			
00037:	00061	XXXXX	00160	----			
00037:	00062	XXXXX	00223	----			
00037:	00063	XXXXX	00000	----			
00037:	00064	XXXXX	00016	----			
00037:	00065	00000	00000	----			
00038:	00066	60XXX			F7	CALL	MVMULT,M,N,L,XHAT,LXHAT,E+
00038:	00067	XXXXX	00001	----			
00038:	00070	XXXXX	00000	----			
00038:	00071	XXXXX	01246	----			
00038:	00072	XXXXX	00000	----			
00038:	00073	XXXXX	00242	----			
00038:	00074	XXXXX	00017	----			
00038:	00075	00000	00000	----			
00039:	00076	60XXX			F8	CALL	VSUB,M,ZEROV,LXHAT,U,E+8,0
00039:	00077	XXXXX	00001	----			
00039:	00100	XXXXX	00254	----			
00039:	00101	XXXXX	00242	----			
00039:	00102	XXXXX	00017	----			
00039:	00103	XXXXX	00020	----			
00039:	00104	00000	00000	----			
00040:							* RESCALING OF XHAT AND U.
00041:	00105	60XXX				CALL	RESCA,N,FXHAT,XHAT,E+10,0
00041:	00106	XXXXX	00000	----			
00041:	00107	XXXXX	00266	----			
00041:	00110	XXXXX	00000	----			
00041:	00111	XXXXX	00022	----			
00041:	00112	00000	00000	----			
00042:	00113	60XXX				CALL	RESCA,M,FU,U,E+11,0
00042:	00114	XXXXX	00001	----			
00042:	00115	XXXXX	00274	----			
00042:	00116	XXXXX	00017	----			
00042:	00117	XXXXX	00023	----			
00042:	00120	00000	00000	----			
00043:	00121	45657	00000	----		J,I	CALG
00044:							* TAG TABLE
00045:	00122		00017		FIX	BSS	15
00046:	00141		00017		GU	BSS	15
00047:	00160		00017		FIXPGU	BSS	15
00048:	00177		00012		CX	BSS	10
00049:	00211		00012		YMCX	BSS	10
00050:	00223		00017		KYCX	BSS	15
00051:	00242		00012		LXHAT	BSS	10
00052:	00254	00000	00000		ZEROV	BSS	10,0
00053:						NAME	FXHAT
00054:						NAME	FU
00055:	00266	00012			FXHAT	DEC	2,2,1,1,1,1
00055:	00267	00004		----			
00055:	00270	00002		----			

APPENDIX C  
(continued)

00055:	00271	00001	---			
00055:	00272	00001	---			
00055:	00273	00001	---			
00056:	00274	00002	---	FU	DEC	1, 4,
00056:	00275	00010	---			
00057:		00000			END	0

```

00001:
00002: *SUBROUTINE MVMULT, PERFORMS
00003: *MATRIX-VECTOR MULTIPLICATION.
00004: *PROGRAMMER: JONAS AGERBERG.
00005: *DATE 9.11.67
00006: *REVISION 17. SEPT. 68
00007: *A MATRIX, A(N,M), (N ROWS, M COLUMNS)
00008: *IS POSTMULTIPLIED BY A VECTOR, X(M).
00009: *THE PRODUCT IS A VECTOR, Y(N).
00010: *ALL ELEMENTS ARE IN SINGLE PRECISION
00011: *ON OVERFLOW ON ANY ELEMENT OPERATION
00012: *THE ELEMENT WILL BE SET TO MAX OR MIN
00013: *FRACTIONAL VALUE (OCT 77777 OR 100000)
00014: *AND CELL 'ERROR' WILL BE INCREMENTED BY
00015: *ONE. THIS CELL CAN BE TESTED BY MAIN
00016: *PROGRAM
00017: *
00018: *CALLING SEQUENCE:
00019: ** L MVMULT
00020: ** ADR N
00021: ** ADR M
00022: ** ADR A
00023: ** ADR X
00024: ** ADR Y
00025: ** ADR ERROR
00026: ** ADR 0
00027: *
00028: *DATA FIELD SHOULD BE DEFINED BY
00029: ** A BSS M.N
00030: ** X BSS M
00031: ** Y BSS N
00032: *
00033: *
00034: *
00035: *PROGRAM STARTS HERE
00036: NAME MVMULT
00037: 01000 00000 REL 0
00038: 00000 00000 SHIFT EQU 0 OR SET AT ASSEMB
00039: 00000 100000 00000 MVMULT ADR, I 0
00040: 00001 51076 00077 STX SAVX
00041: 00002 145776 00000 LA, I MVMULT FETCH N
00042: 00003 161062 00065 STA N
00043: 00004 20100 TCA
00044: 00005 26400 SSP SET MINUS N
00045: 00006 161061 00067 STA MNX FOR INDEX
00046: 00007 71771 00000 AOM MVMULT
00047: 00010 145770 00000 LA, I MVMULT FETCH M
00048: 00011 161055 00066 STA M
00049: 00012 20100 TCA
00050: 00013 26400 SSP SET MINUS M
00051: 00014 161054 00070 STA MMX FOR INDEX
00052: 00015 141763 00000 LA MVMULT FETCH STRING
00053: 00016 26400 SSP ADDRESSES AND
00054: 00017 26500 EX STORE THEM
00055: 00020 142001 00001 LA, X 1
00056: 00021 151045 00066 A M
00057: 00022 161047 00071 STA ADRA
00058: 00023 142002 00002 LA, X 2
00059: 00024 151042 00066 A M
00060: 00025 161045 00072 STA ADRX
00061: 00026 142003 00003 LA, X 3
00062: 00027 151036 00065 A N
00063: 00030 161043 00073 STA ADRY

```





```

00001: *
00002: 01000 00000 REL 0
00003: NAME VSUB
00004: NAME VADD
00005: * PERFORMS VECTOR ADD/SUBTRACT, C=A+B OR
00006: * C=A-B IN SINGLE PRECISION
00007: * DIMENSION OF VECTORS = DIM
00008: * IF AN ELEMENT OF C OVERFLOWS IT IS SET
00009: * TO MAX (OR MIN) FRACTIONAL VALUE
00010: * ON EACH OVERFLOW (ERR) IS INCREMENTED BY ONE
00011: * NO ERROR EXITS
00012: * VSUB USES VADD AFTER FIXING
00013: * PROGRAMMER :J AGERBERG
00014: * DATE 12 SEPT 68
00015: * REVISED
00016: *
00017: * FORTRAN CALL :
00018: ** CALL VADD (DIM, A, B, C, ERR)
00019: * OR CALL VSUB (DIM, A, B, C, ERR)
00020: *
00021: * ASSEMBLER CALL :
00022: * L VADD (OR VSUB)
00023: * ADR DIM
00024: * ADR A
00025: * ADR B
00026: * ADR C
00027: * ADR ERR
00028: * ADR 0
00029: *
00030: *
00031: *
00032: 00000 100000 00000 VSUB ADR, I 0
00033: 00001 141777 00000 LA *-1 FIX RETURN
00034: 00002 161004 00006 STA VADD ADDRESS
00035: 00003 141033 00036 LA GP FIX SUBTRACT
00036: 00004 101057 00063 OR SUBMSK INSTRUCTION
00037: 00005 41004 00011 J ADD
00038: 00006 100000 00000 VADD ADR, I 0
00039: 00007 141027 00036 LA GP FIX ADD
00040: 00010 131052 00062 AND ADDMSK INSTR.
00041: 00011 161025 00036 ADD STA GP
00042: 00012 51054 00066 STX SAVX
00043: 00013 141773 00006 LA VADD
00044: 00014 26400 SSP REMOVE INDBIT
00045: 00015 26500 EX PT TO L+1
00046: 00016 142001 00001 LA, X 1 FETCH ARG
00047: 00017 161036 00055 STA ABASE ADDRESSES
00048: 00020 142002 00002 LA, X 2
00049: 00021 161035 00056 STA BBASE
00050: 00022 142003 00003 LA, X 3
00051: 00023 161034 00057 STA CBASE
00052: 00024 142004 00004 LA, X 4
00053: 00025 161033 00060 STA ADRERR
00054: 00026 22006 00006 ICX 6
00055: 00027 51032 00061 STX EXIT
00056: 00030 145756 00006 LA, I VADD FETCH DIM
00057: 00031 25500 EX (XR) - DIM
00058: 00032 22777 00001 MORE DCX 1
00059: 00033 27417 SKU
00060: 00034 41030 00064 J OUT
00061: 00035 147020 00055 LA, IX ABASE
00062: 00036 157020 00056 OP A, IX BBASE ADD/SUBTR
00063: 00037 167020 00057 STA, IX CBASE

```

APPENDIX C  
(continued)

00064:	00040	27401		SG		OVFL?
00065:	00041	41771	00032	J	MORE	NO
00066:	00042	24040		SKP		YES, OVFL NEG?
00067:	00043	41004	00047	J	PGS	NO, GOTO PGS
00068:	00044	26740		CLR		YES, SET
00069:	00045	26440		SSN		(AR) = -1
00070:	00046	41004	00052	J	STOVF	
00071:	00047	26740		PGS CLR		SET (AR)
00072:	00050	20200		OCA		MAX PGS
00073:	00051	26400		SSP		
00074:	00052	167005	00057	STOVF STA, IX	CBASE	STORE MAX/MIN
00075:	00053	75005	00060	AGM, I	ADRERR	INCR. ERRCOUNT
00076:	00054	41756	00032	J	MORE	
00077:	00055	00000	00000	ABASE BSS	1, 0	
00078:	00056	00000	00000	BBASE BSS	1, 0	
00079:	00057	00000	00000	CBASE BSS	1, 0	
00080:	00060	00000	00000	ADRERR BSS	1, 0	
00081:	00061	00000	00000	EXIT BSS	1, 0	
00082:	00062	157777		ADDMSK OCT	157777	
00083:	00063	170000		SUBMSK OCT	170000	
00084:	00064	53002	00066	GUT LX	SAVX	
00085:	00065	45774	00061	J, I	EXIT	
00086:	00066	00000	00000	SAVX BSS	1, 0	
00087:		00000		END	0	

```

00001: * SUBROUTINE RESCA.
00002: * PERFORMS
00003: * VECTOR A IN FRACTIONAL MULT WITH
00004: * VECTOR SCALE IN INTEGER.
00005: * RESULT IN FRACTIONAL PLACED IN
00006: * VECTOR A.
00007: * IF OVERFLOW OCCUR ,AGM E,MAX VALUE
00008: * WITH SIGN STORED.
00009: * CALLING SEQ.
00010: * CALL RESCA,N,SCALE,A,E,0
00011: * SCALE N-DIM VECTOR IN INTEGER.
00012: * A N-DIM VECTOR IN FRACTIONAL
00013: * E NAME OF OF ERROR CELL.
00014: *
00015: *
00016: 01000          00000          REL          0
00017:                                NAME          RESCA
00018: 00000 100000 00000          RESCA  ADDR,I    0
00019: 00001  53777 00000          LX           *- 1
00020: 00002 142000 00000          LA,X          0          LOAD FIRST ARG
00021: 00003   20100          TCA
00022: 00004 161046 00052          STA          COUNT
00023: 00005   26500          EX                      ADR TO SCALE
00024: 00006   26400          SSP                      STORED IN TAS
00025: 00007   26500          EX
00026: 00010 142001 00001          LA,X          1
00027: 00011 161040 00051          STA          TAS
00028: 00012 142002 00002          LA,X          2          ADR TO A
00029: 00013 161035 00050          STA          TAA          STORED IN TAA
00030: * MULTIPLICATION
00031: 00014  53037 00053          LX          ZERO
00032: 00015 147033 00050          LP          LA,IX          TAA
00033: 00016  37033 00051          M,IX          TAS
00034: 00017 161030 00047          STA          TEMP
00035: 00020  26117 00017          ALD          15
00036: 00021  27416          SNG
00037: 00022  41011 00033          J          ERR
00038: 00023 167025 00050          AE          STA,IX          TAA
00039: 00024  22001 00001          ICX          1
00040: 00025  71025 00052          AGM          COUNT
00041: 00026  41767 00015          J          LP
00042: 00027 141751 00000          LA          RESCA          EXIT
00043: 00030   26400          SSP
00044: 00031   26500          EX
00045: 00032  42005 00005          J,X          5
00046: 00033  51021 00054          ERR          STX          TMP2
00047: 00034  53744 00000          LX          RESCA
00048: 00035  72003 00003          AGM,X          3          4TH ARG
00049: 00036 141011 00047          LA          TEMP
00050: 00037 151014 00053          A          ZERO
00051: 00040 141006 00046          LA          MAX
00052: 00041  53013 00054          LX          TMP2
00053: 00042  27402          SM
00054: 00043  41760 00023          J          AE
00055: 00044  20100          TCA
00056: 00045  41756 00023          J          AE
00057: 00046  77777          MAX          OCT          77777
00058: 00047  00000 00000          TEMP          BSS          1,0
00059: 00050  00000 00000          TAA          BSS          1,0          ADR TO A
00060: 00051  00000 00000          TAS          BSS          1,0          ADT TO SCALE
00061: 00052  00000 00000          COUNT          BSS          1,0          NO OF ELEMENTS
00062: 00053  00000          ZERO          DEC          0
00063: 00054  00000 00000          TMP2          BSS          1,0

```

00064:

00000

END

0

## APPENDIX D

Consider the system

$$\mathbf{x}(t+1) = \phi^{**} \mathbf{x}(t) + \Gamma \mathbf{u}(t) + \mathbf{e}_1(t)$$

$$y(t) = \theta \mathbf{x}(t) + \mathbf{e}_2(t) \quad (1)$$

where  $\phi^{**}$  is the disturbed matrix  $\phi$ . The Kalman filter is

$$\hat{\mathbf{x}}(t+1) = \phi \hat{\mathbf{x}}(t) + \Gamma \mathbf{u}(t) + K[y(t) - \theta \hat{\mathbf{x}}(t)] \quad (2)$$

Equations (1) and (2) give

$$\mathbf{x}(t+1) - \hat{\mathbf{x}}(t+1) = (\phi - K\theta)(\mathbf{x}(t) - \hat{\mathbf{x}}(t)) + (\phi^{**} - \phi)\mathbf{x}(t) + \mathbf{e}_1(t) - K\mathbf{e}_2(t)$$

The deterministic part of the reconstruction error  $\tilde{\mathbf{x}}(t) = \mathbf{x}(t) - \hat{\mathbf{x}}(t)$  then is

$$\tilde{\mathbf{x}}(t+1) = (\phi - K\theta)\tilde{\mathbf{x}}(t) + (\phi^{**} - \phi)\mathbf{x}(t)$$

In steady state we get

$$\tilde{\mathbf{x}}_0 = (\mathbf{I} - \phi + K\theta)^{-1} (\phi^{**} - \phi) \mathbf{x}_0 \quad (3)$$

Using the steady state value of the state vector corresponding to the undisturbed system equation (3) will give an estimate of the true reconstruction error.